The 4<sup>th</sup> European sCO<sub>2</sub> Conference for Energy Systems March 22-26, 2021, Prague, Czech Republic

# Sofia – sCO2 facility for Supercritical Brayton Cycle Research

Otakar Frýbort, Karel Dočkal, Petr Vlček, Petr Hájek Jr., Tomáš Melichar, Miroslav Kapic, Antonín živný, Marek Pátý, Aleš Macálka, Vilém hanzal, Petr Hájek

23.3.2021







#### Content

- Context, project sCO2 Efekt and Sofia facility
- Layout of the Sofia facility
- Main components of the Sofia facility and its parameters
- Location and experimental plan
- Conclusion





# Context, project sCO2 Efektand Sofia facility

- sCO2 Efekt (sCO2 Effect) is a research project supported by TAČR (Czech Technological Agency)
- Sofia is the name of the experimental facility to be developed in the frame of sCO2 Efekt project
- Project partners CVR, UJV, Doosan
- Main goal of the project development of the conceptual design of the effective Energy Storage System and verification of its key components





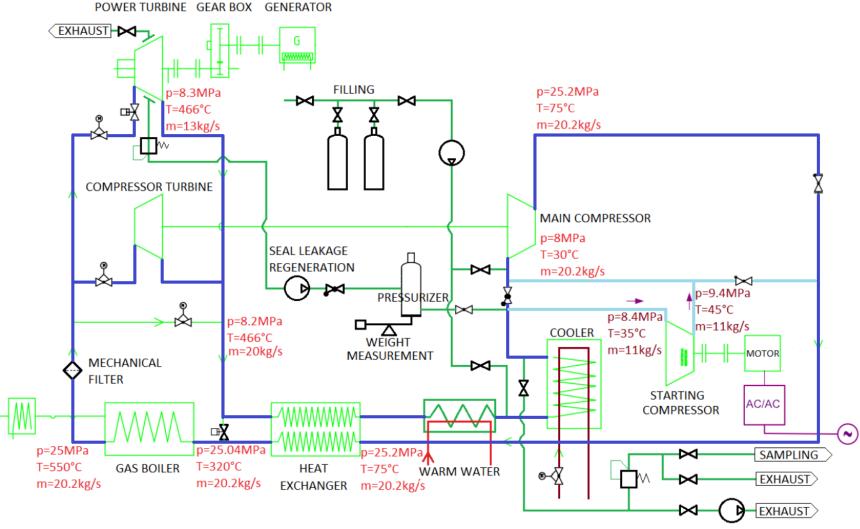
# Context, project sCO2 Efekt and Sofia facility

- The Energy Storage System being developed is based on the TES (Thermal Energy Storage) principle
- Aluminium Silicon eutectic alloy AlSi12 is used as a thermal storage media
- Most of the heat is stored in the form of latent heat of AlSi12 alloy
- sCO2 power cycle is used for the reverse power production
- The details about the Storage tank are included in another presentation –
   "Thermal design of latent heat thermal energy storage facility with supercritical CO2" to be introduced by Tomas Melichar on Wednesday at 11:00





### Layout of the Sofia facility







# Main components of the Sofia facility and its parameters

- Compander
- Starting compressor
- Power turbine
- Pressurizer
- Heat source, heat exchanger and coolers





### Compander

- Compander is the main compressor driven by its centrifugal turbine
- It is not equipped with a motor
- During the start-up procedure, the main compressor works in series with the starting compressor
- The shaft is mounted in two radial and one thrust gas bearings
- Main parameters measured during the operation are:
  - Rotation speed, rotor vibrations, global vibrations, internal pressure and axial force



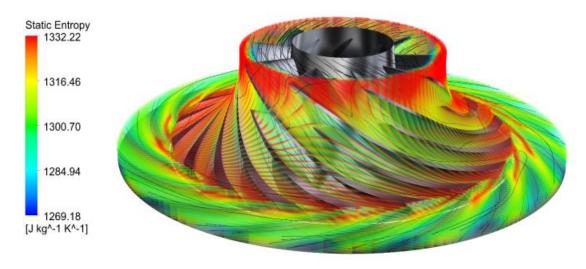


## Compander – main compressor

- Main parameters of the compressor at the design point are:
  - Pressure inlet
  - Pressure outlet
  - Inlet temperature
  - Mass flow rate
  - Rotational speed
  - Total efficiency
  - Mach number
  - Axial force



- 25.5 MPa
- 30 °C
- 20.9 kg/s
- 68 000 rpm
- 78 %
- < 0.93
- -2.26 kN



Compressor entropy and streamlines visualization





# **Compander – driving turbine**

Main parameters of the turbine at the design point are:

		4
<ul> <li>Pressur</li> </ul>	$\sim$ 10	-
	<b>—</b> 1111	
1 1 ( / ( ) ( ) ( ) ( )		

Pressure outlet

Inlet temperature

Mass flow rate

Rotational speed

Total efficiency

B2B rel. Mach number

Shaft power

Axial force

24 MPa

8.5 MPa

550 °C

6.19 kg/s

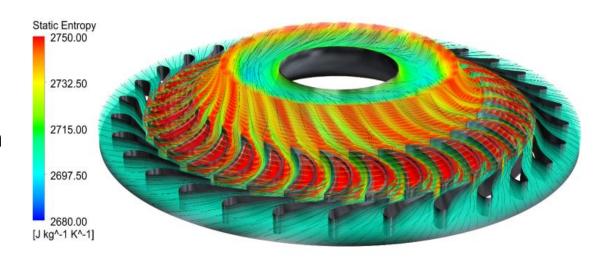
68 000 rpm

79 %

< 1.43

~ 650 kW

2.37 kN



Turbine entropy and streamlines visualization





#### **Compander status**

- CFD analysis of the comperssor and turbine were performed
- Internal seals and axial forces were optimized
- The axial force needs to be measured and controlled during operation, as the thrust gas bearing was designed to handle the force <300N</li>
- Stress analyses of the compressor pressure vessel are ongoing
- The detailed design is currently being completed and fabrication begins



Compander 3d model



# **Starting compressor**

- The compressor will be used during the start-up operation only
- It is being designed as a two stage compressor
- The conceptual studdy of the compressor was performed
- It will be hermetical encapsuled motor driven compressor mounted in gas bearings.
- Design parameters of the compressor are summarized in the following table

Design pressure	10 MPa
Nominal pressure inlet	8.4MPa
Inlet pressure range	5 – 9 MPa
Nominal pressure outlet	9.4MPa
Nominal mass flow rate	11 kg/s
Nominal inlet temperature	35°C
Inlet temperature range	20-40°C
Nominal power of the motor	55 kW
Speed control	Frequency converter
Nominal speed	40 000 rpm





## Power turbine (PT)

- The power turbine is designed as the axial one
- The axial type was selected in an affort to get a first experience and operational data of sCO2 axial turbine.
- PT together with the budget limit defined the power of Sofia facility
- Design parameters of the PT are:

Design pressure	26.5 MPa
Nominal pressure inlet	25 MPa
Nominal pressure outlet	8.5 MPa
Nominal mass flow rate	13 kg/s
Nominal inlet temperature	550 °C
Inlet temperature range	20-40 °C
Nominal power of the turbine	1050 kW
Nominal speed	20 000 rpm
Number of blade stages	5

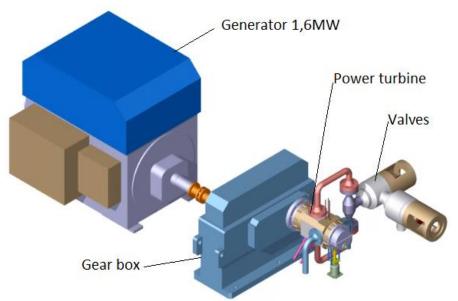




# Power turbine (PT)

- The power turbine is located directly on the gear box
- The turbine is being developed and will be delivered by Doosan Škoda Power including the power generator
- Nominal power of the gear box and generator 1.6 MWe
- The turbine is equipped with the dry gas seals which are connected with the leakage
- The turbine workspace is created by 5 axial stages
- The length of the first blade si about four milimeters
- PT is designed with common oil lubricated bearings
- Detail design of the turbine is beeing finished





#### **Pressurizer**

- Main goals of the pressurizer are:
  - sCO2 storage
  - Keeping of the bottom pressure on the required level
  - Conditioning of the cycle at the beginning of start-up procedure
- The pressurizer enables to store up to 400 kg of sCO2
- Its content is controlled by temperature
- The vessel is equipped with electrical heating rods and two independent cooling systems

• The content of sCO2 is determined by weight monitoring





3d model of the pressurizer

### Heat source, heat exchanger and coolers

The heat source parameters were defined as follows

Design pressure	26.5 MPa
Nominal Pressure	25 MPa
Design temperature	565 ° C
Nominal inlet temperature	320 ° C
Nominal outlet temperature	550 ° C
Nominal CO2 mass flow rate	22 kg/s
Heat power of ignition burner	100 kW
Maximum heating power supplied to sCO2	6000 kW
Total heating power	6850 kW

- Gas boiler was expected to be a heat source, but another solution is currently also being solved
- Heat exchanger will be the PCHX one
- The precooler and cooler (sCO2/water) will be of the plate HX type
- Specific parameters of the cooling system are included in the paper.



## Location and experimental plan

- The original plan is to build and operate the Sofia facility at UJV (and CVR) area
- Czech power producer and operator ČEZ expressed the interest in the implementation of the Sofia facility at its power and heating plant
- Feasibility studdy of this modification is intensively addressed
- The goal of the project is to test the compressors and turbines
- The location of the facility at the power plant area will enable long term operation even without further financing (which is expected)



#### **Conclusions**

- Detail design of the compander and power turbine is almost finished. The fabrication of the facility as well as the tenders starts at 6/2021
- The facility will be assembled at the end of 2022
- First operational experience will be gained during 2023, when the project Efekt will be finished
- Further modification of the Sofia facility to recompression cycle is discussed
- Utilization of the facility within EU projects is expected
- Any collaboration and/or testing of the cycle components is possible and very welcome



# Thank you for your attention!

#### Otakar Frybort Research Centre Rez otakar.frybort@cvrez.cz

This works was supported by TACR THETA2, project no. TK02030059 (Efekt).





