

The 4th European sCO₂ Conference for Energy Systems

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Thermal Efficiency Gains Enabled by Using Supercritical CO₂ Mixtures In Concentrated Solar Power Applications

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Supercritical CARBON dioxide/Alternative fluids
Blends for Efficiency Upgrade of Solar power plant



This project has received funding from the European Union's Horizon 2020
research and innovation programme under grant agreement No. 814985

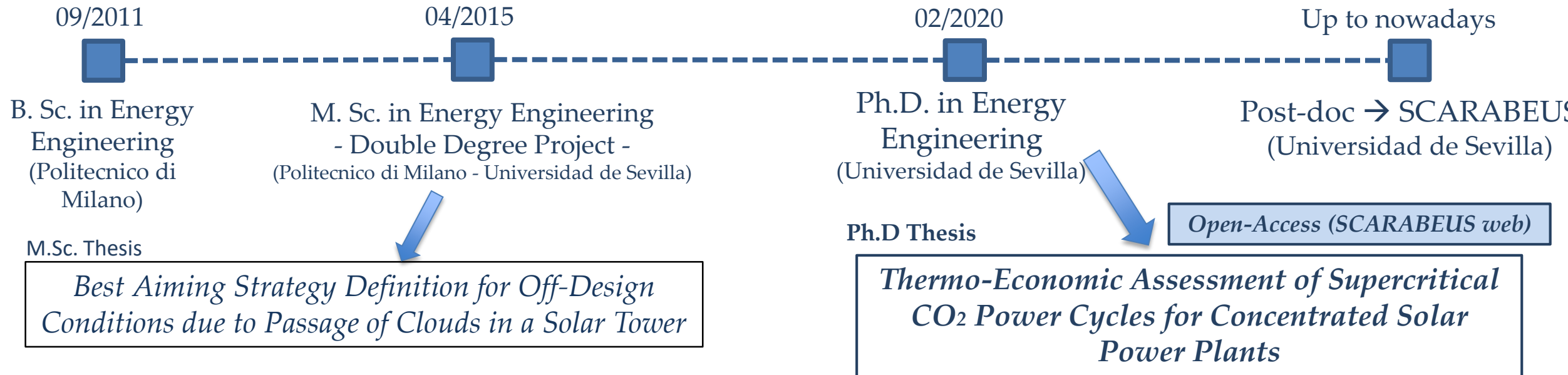


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Current position: Post-doc fellow at Thermal Power Group (Universidad de Sevilla)

Main Research interests: sCO₂ power cycles, Concentrated Solar Power, Power cycles modelling and simulation, Renewable Energy, Heat transfer, Heat Exchangers.





- ❑ 1. INTRODUCTION TO SCARABEUS CONCEPT
- ❑ 2. COMPUTATIONAL ENVIRONMENT & AIM OF THE STUDY
- ❑ 3. ANALYSIS OF THE RESULTS
 - BEST COMBINATION BETWEEN sCO₂ BLEND AND CYCLE LAYOUT
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- ❑ 4. CONCLUSIONS
- ❑ 5. Q&A



Combination of sCO₂ and Concentrated Solar Power (Solar Tower)

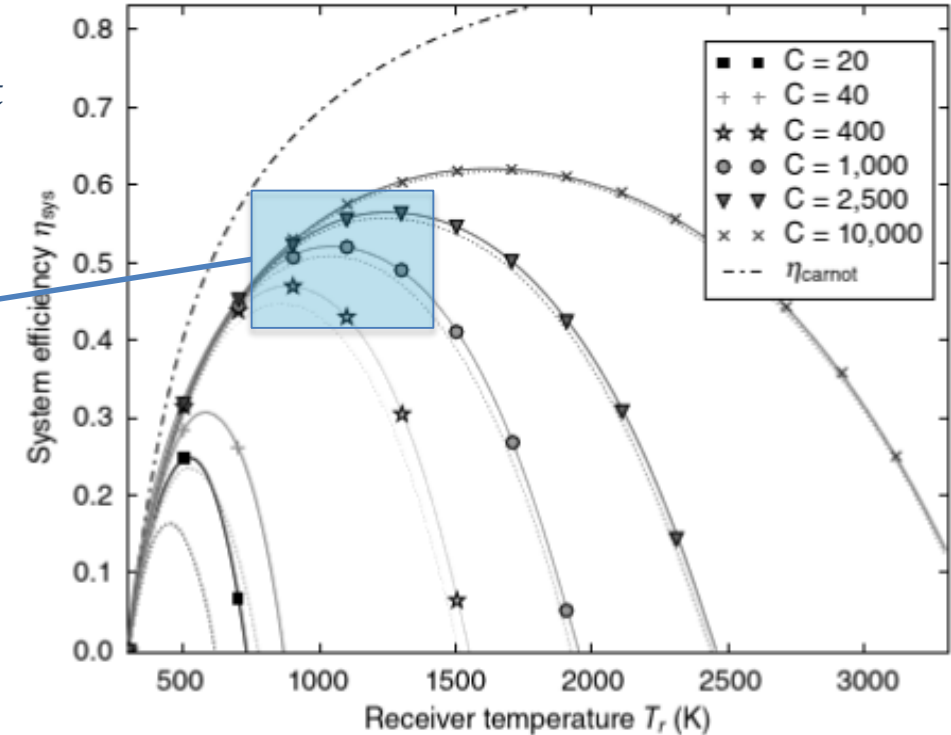
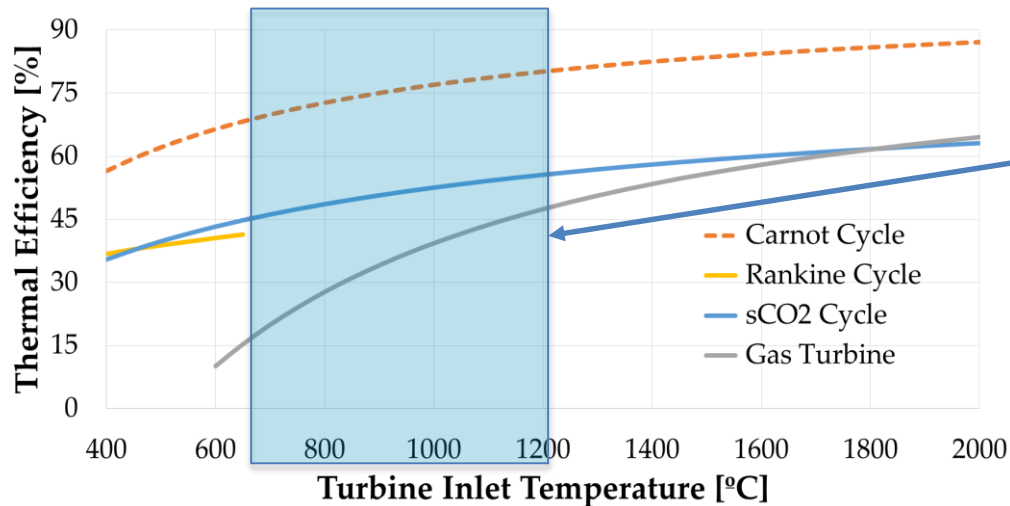
$$\eta_{rec} \propto C, T_{rec}$$

Concentration factor = 1000 \rightarrow η_{sys} optimised by $T = 650 \div 1150^{\circ}\text{C}$

$$\eta_{sys} = \eta_{rec} * \eta_{pb}$$

sCO₂ cycles are promising in this particular range of temperatures

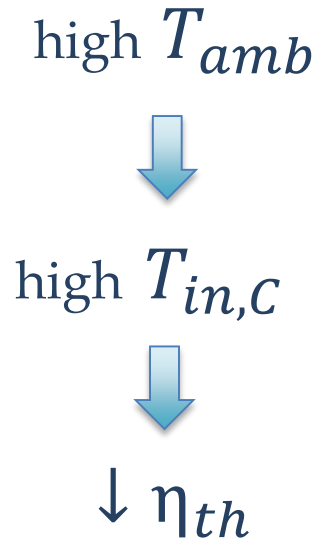
Main Limitation: Need for Compressor Inlet Temperature close to Critical Point





Combination of sCO₂ and Concentrated Solar Power (Solar Tower)

Main Limitation: Compressor Inlet Temperature close to Critical Point difficult to attain in usual CSP locations



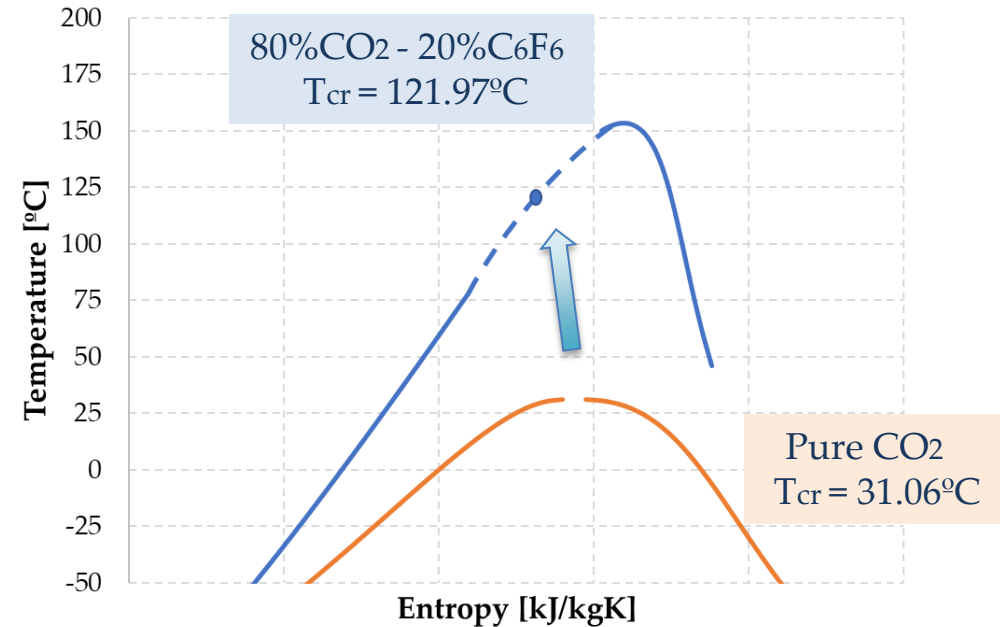
Cycle layout (Turbine Inlet: 300bar, 750°C)	η_{th} @ 32°C [%]	η_{th} @ 50°C [%]	$\Delta\eta_{th}$ [pp]
Simple Recuperated	46.2	43.5	2.7
Precompression	50.0	46.9	3.1
Recompression	51.4	47.9	3.5
Recompr.+IC+RH	53.0	49.1	3.9
Partial Cooling	51.6	48.3	3.3
Partial Cooling+RH	53.9	48.9	5.0
Double Reheated	54	44.3	9.7
Schroder - Turner	49.0	45.3	3.7
Modified Allam	45.6	43.5	2.1
Transcritical CO ₂	48.3 (15°C)	-	-



SCARABEUS project → possible solution to overcome this limitation

Supercritical CARbon dioxide/Alternative fluids Blends for Efficiency Upgrade of Solar power plant

- ❑ Main Concept:
Addition of dopants to increase critical temperature, enabling fluid condensation at high ambient temperatures hence increasing cycle performance
- ❑ Main Objectives:
 - Thermal efficiency > 50% ($T_{\min} \geq 50^{\circ}\text{C}$)
 - CAPEX and OPEX Reduced by 30% and 35% respectively (compared with SoA steam-based CSP)
- ❑ Consortium: five Universities, four companies



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Baker Hughes



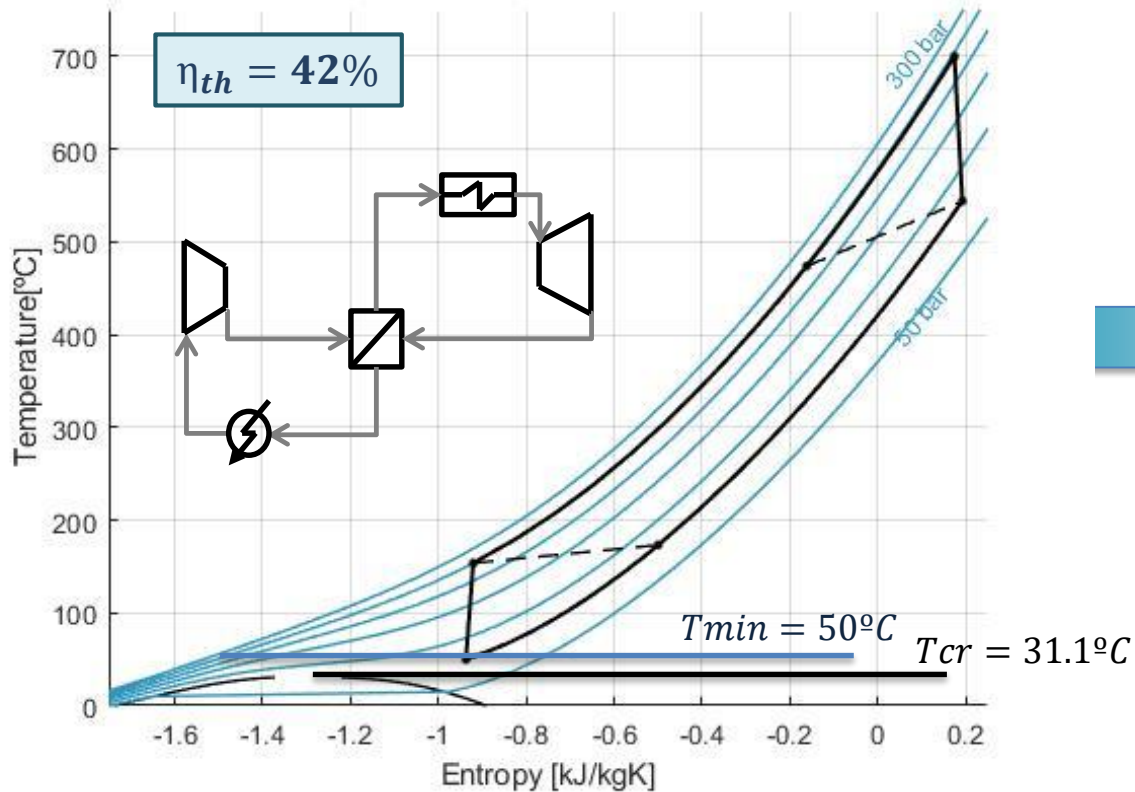
ABENGOA

Quantis

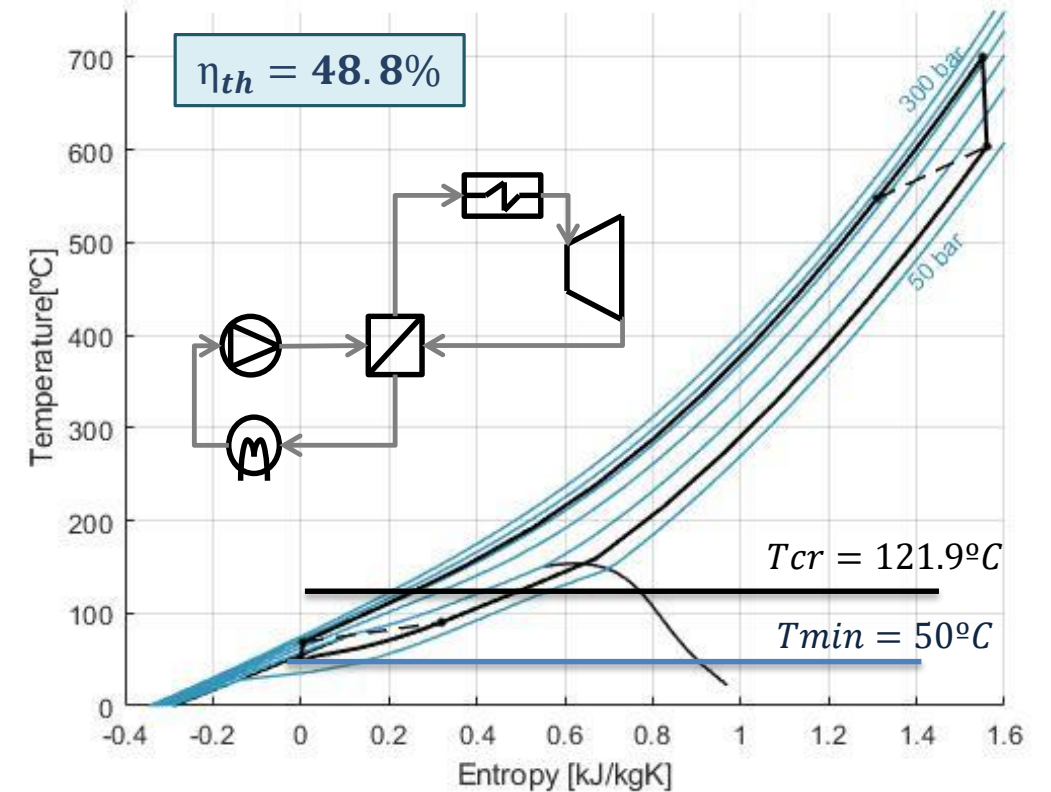


CO₂ blended with dopants → Rise in Critical Temperature → Condensation at high ambient temperature is enabled

Recuperated Brayton (pure CO₂)



Recuperated Rankine (80% CO₂ – 20% C₆F₆)





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Scope of Present Study → explore the actual potential of different sCO₂-based blends

- ❑ **Two Candidate Dopants:** Hexafluorobenzene (C₆F₆) and Titanium Tetrachloride (TiCl₄)

	Health Hazard	Flammability	Chemical Reactivity	Special Hazard
sCO ₂	2	0	0	SA
C ₆ F ₆	1	3	0	-
TiCl ₄	3	0	2	W

Due to safety issues, other candidate dopants are currently being investigated by SCARABEUS Consortium. This does not undermine the conclusions of this work.

- ❑ **Two Candidate Cycles:** *Recuperated Rankine* and *Precompression* cycles

- ❑ **Reference Case:** 100MW (Gross Power)

T _{min} [°C]	TIT [°C]	P _{max} [bar]	η _{is} [%] Pump/Turb/Compr
50	550/700	250/300	88 / 93 / 89
ΔT _{min} in HX [°C]	ΔP _{HEATER} [%]	ΔP _{COND} [%]	ΔP _{REC} [%] Low P / High P
5	1.5	0	1 / 1.5

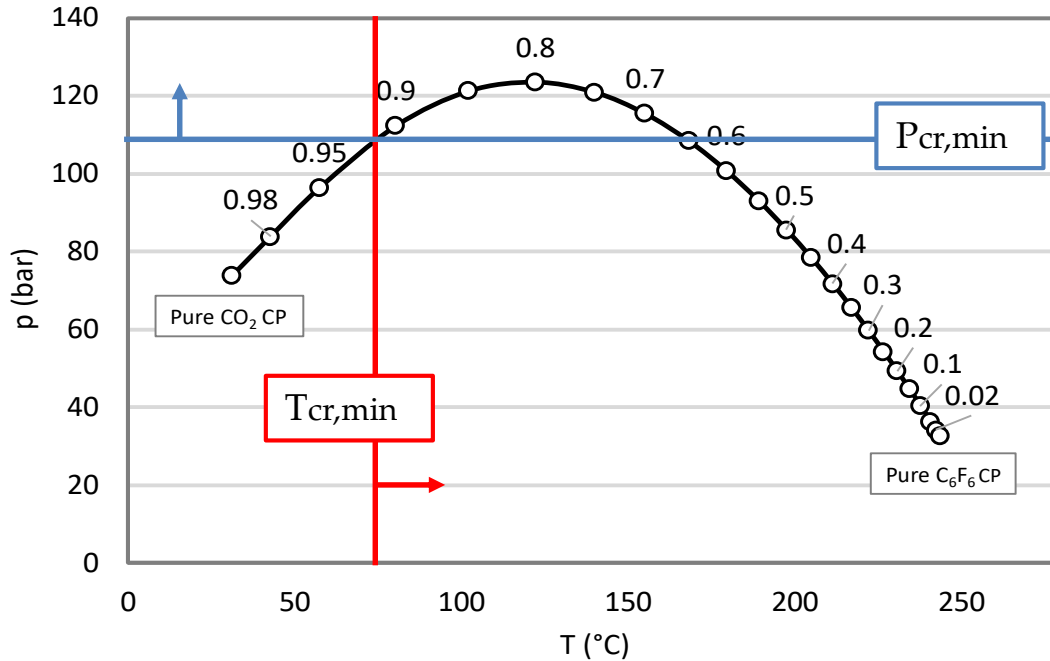
SCARABEUS Reference Case
 T_{min}=50°C – TIT=700°C
 P_{max}=250bar



Seven candidate blends, defined by molar fraction of dopant

Dopant molar fraction selection criterion

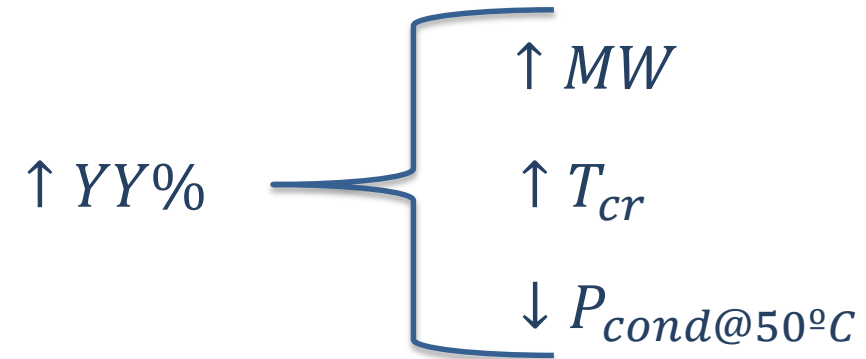
$T_{cr} - T_{min} \geq 30^\circ\text{C} \rightarrow T_{cr} \geq 80^\circ\text{C}$ (SCARABEUS case)



Mixture Code: DXCYY (applicable to binary mixtures only)

- DX: Dopant (D1=C₆F₆, D2=TiCl₄)
- CYY: YY% molar fraction of DX in mixture

Blend	Composition [% molar]	MW [g/mol]	T_{cr} [°C]	P_{cr} [bar]	P_{cond} [bar]
D1C10	CO ₂ -C ₆ F ₆ [90-10]	58.21	80.28	112.4	83.51
D1C15	CO ₂ -C ₆ F ₆ [85-15]	65.32	102.1	121.3	77.52
D1C20	CO ₂ -C ₆ F ₆ [80-20]	72.42	121.9	123.6	71.83
D1C25	CO ₂ -C ₆ F ₆ [75-25]	79.52	139.8	121.1	66.36
D2C15	CO ₂ -TiCl ₄ [85-15]	65.86	93.76	190.9	99.53
D2C20	CO ₂ -TiCl ₄ [80-20]	73.15	149.6	243.7	97.63
D2C25	CO ₂ -TiCl ₄ [75-25]	80.43	192.0	247.1	94.52





Initial Screening: Part-flow configurations (*Recompression* and *Partial Cooling*) have also been explored but dismissed due to poor thermal performances or poor adaptability to liquid-phase compression when sCO₂ blends are employed.

More info in:

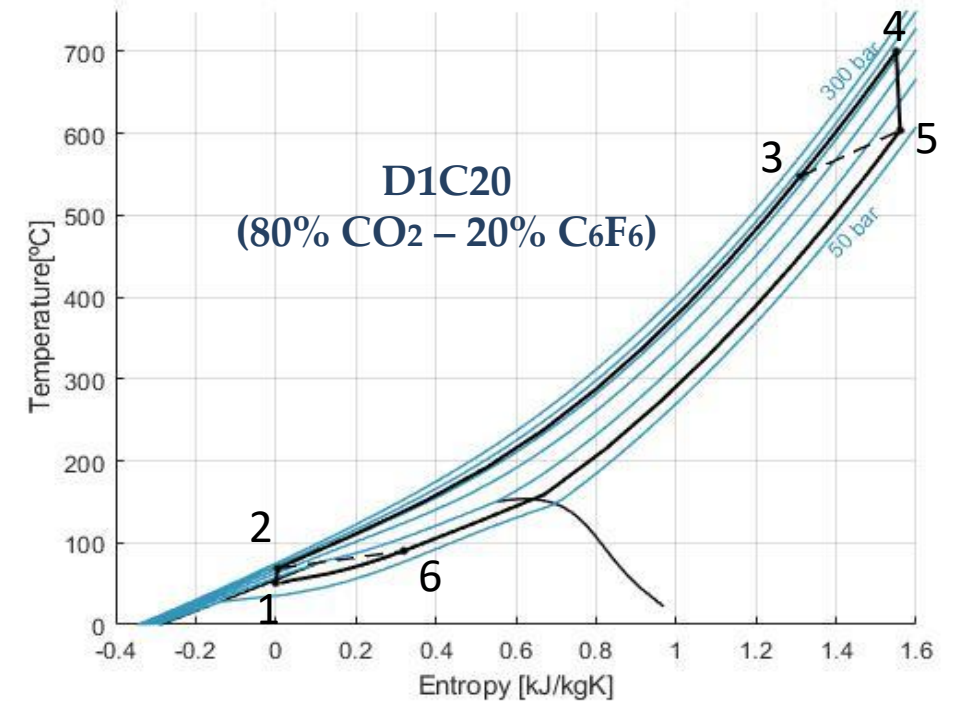
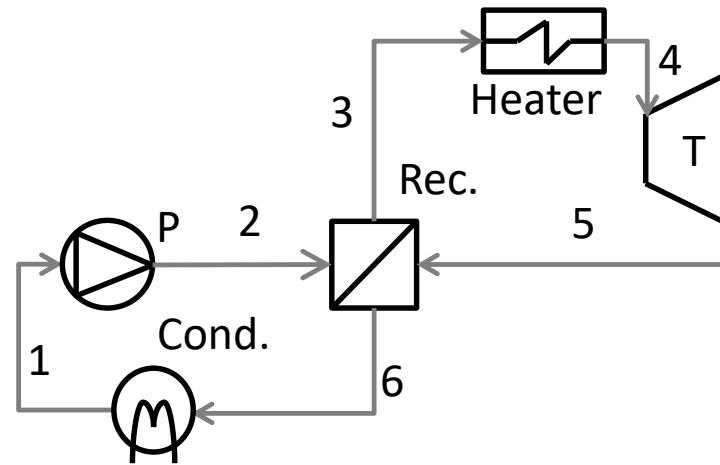
F. Crespi, G.S. Martínez, P.R. de Arriba, D. Sánchez, F. Jiménez-Espadafor, *Influence of Working Fluid Composition on the Optimum Characteristics of Blended Supercritical Carbon Dioxide Cycles*, **ASME Turbo Expo 2021: Turbomachinery Technical Conference and Expositions** (June 2021)



1) Recuperated Rankine

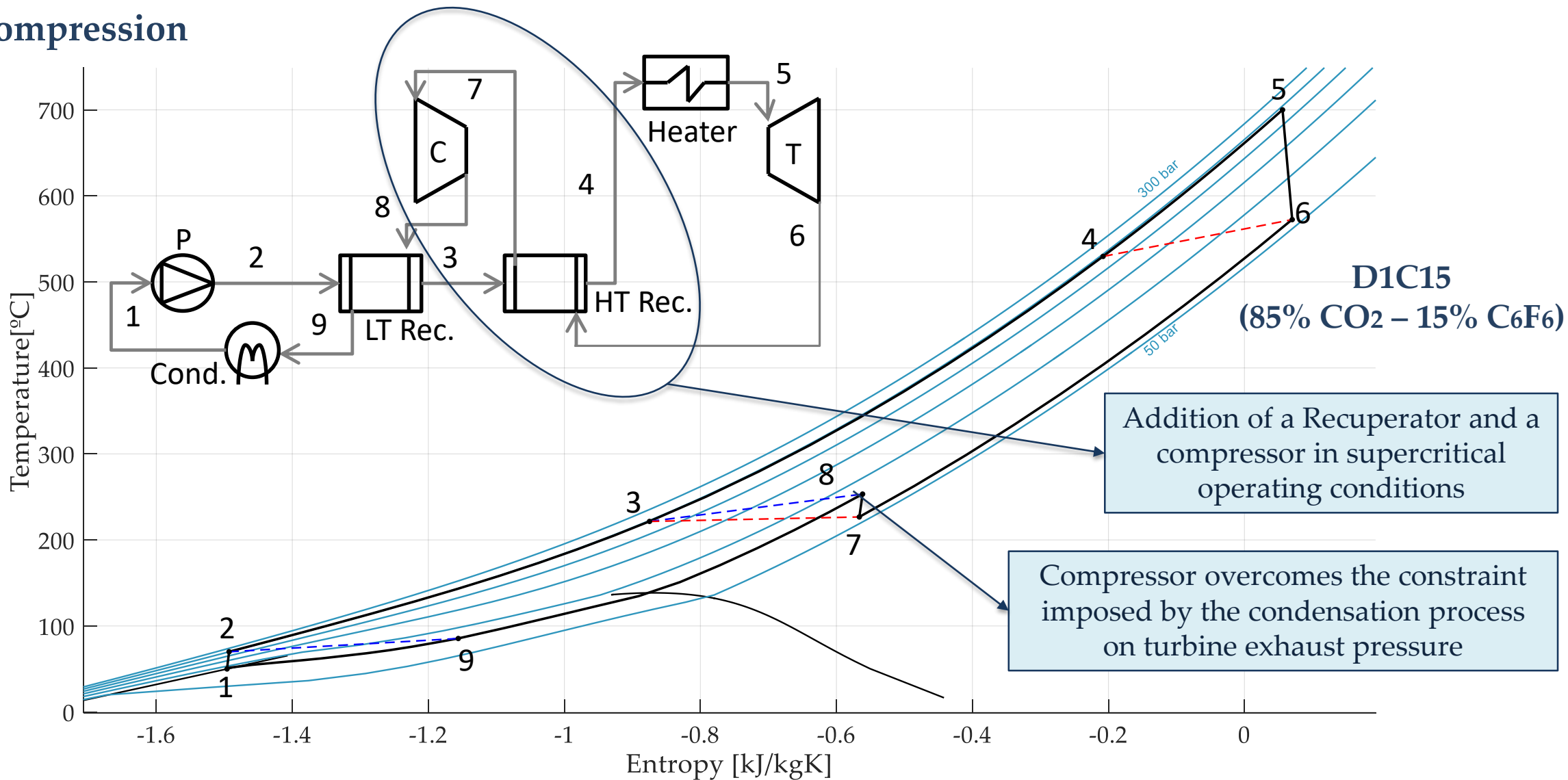
Very Simple Layout

Strong Dependence of Turbine Exhaust on Condensation pressure, i.e. minimum temperature





2) Precompression





a) Calculation of Thermodynamic Properties of the Working Fluid



+ Screening, Evaluation and Selection of Candidate dopants



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LEAP

Laboratorio Energia e Ambiente Piacenza

b) Cycle modeling and simulation



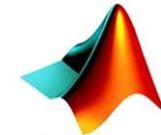
THERMOFLOW

Thermoflex 29
(released on April 2020)



New *User defined fluid* tool,
specifically developed for
SCARABEUS project

c) Post processing

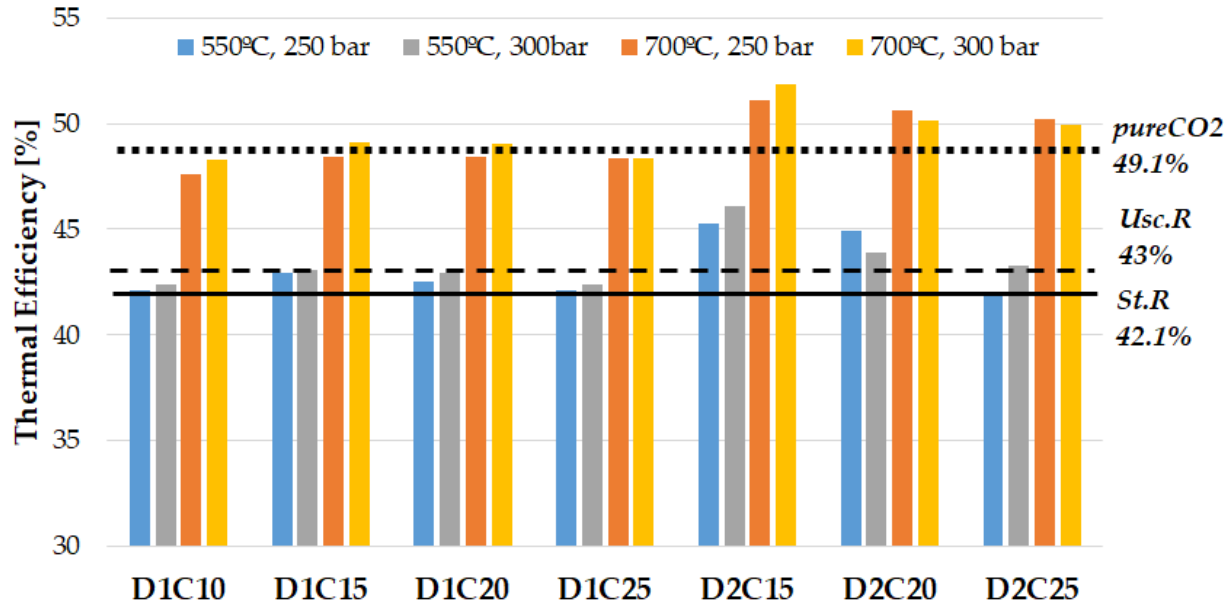


MATLAB

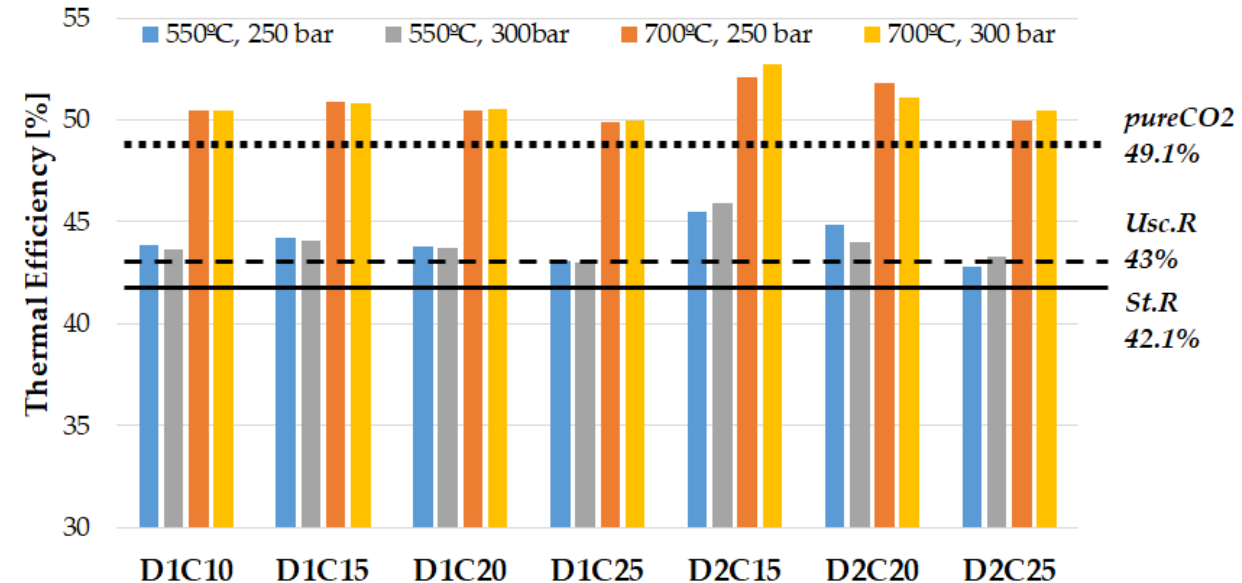




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Recuperated Rankine

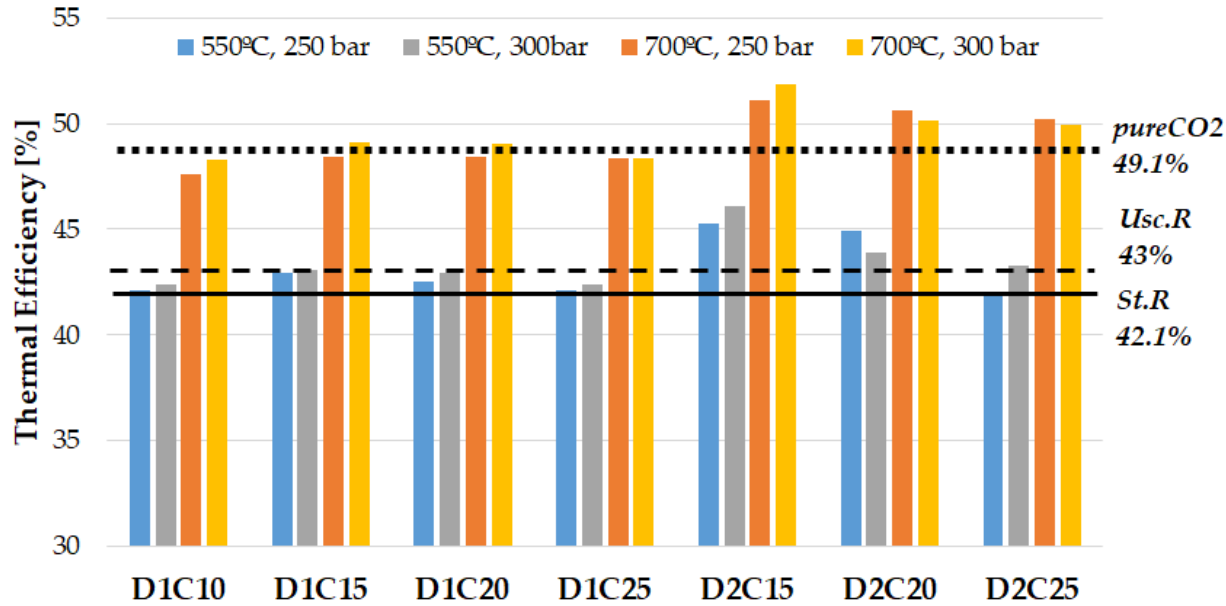


Precompression

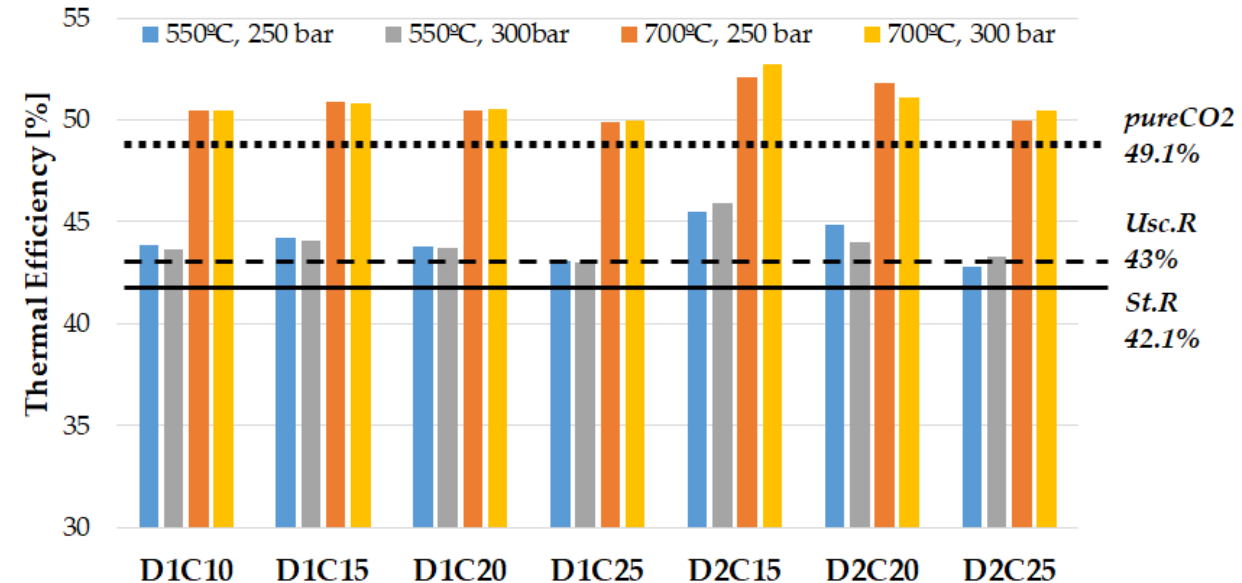
- Thermal efficiency strongly depends on TIT (slightly on Pmax)
- Always enabling η_{th} higher than Standard Rankine (St. R)
- D2C15 always outperforming the other blends
- *Precompression* always enables η_{th} higher than Ultrasupercritical Rankine (Usc. R)

- $TIT \geq 700^\circ C$, *Precompression* with D1 $\eta_{th} > 50\%$
- $TIT \geq 700^\circ C$, D2 always $\eta_{th} > 50\%$

SCARABEUS Objective Achieved!



Recuperated Rankine



Precompression

Cycle layout (Turbine Inlet: 300bar, 750°C)	η_{th} @ 32°C [%]	η_{th} @ 50°C [%]	$\Delta\eta_{th}$ [pp]
Simple Recuperated	46.2	43.5	2.7
Precompression	50.0	46.9	3.1

Pure sCO₂

- D1C15 and D2C15 *Rec. Rankine* lead to 5.5 pp and 8pp higher η_{th} than pure sCO₂ *Rec. Brayton* (with lower TIT)
- D1C15 and D2C15 *Precompression* lead to 3.5pp and 5pp higher η_{th} than pure sCO₂ *Precompression* (with lower TIT)
- Blends outperform pure sCO₂ cycles with similar boundary conditions



Thermal efficiency gains enabled by Precompression with respect to Recuperated Rankine

$$\Delta\eta_{th} = \eta_{th,PRECOMPR.} - \eta_{th,REC.RANKINE}$$

	550°C/250bar	550°C/300bar	700°C/250bar	700°C/300bar
D1C10	1.76	1.28	2.85	2.14
D1C15	1.25	1.04	2.45	1.68
D1C20	1.27	0.81	2.07	1.53
D1C25	0.99	0.63	1.53	1.59
D2C15	0.25	-0.18	0.97	0.84
D2C20	-0.05	0.12	1.15	0.96
D2C25	0.87	0.01	-0.22	0.46

Compressor overcomes the constraint imposed by the condensation process on turbine exhaust pressure

Higher cycle complexity, but with the interesting addition of one degree of freedom (optimisation)

$\Delta\eta_{th}$ strongly affected by the nature and molar fraction of dopant



Precompression exploits the potential of sCO₂-C₆F₆

Recuperated Rankine exploits the potential of sCO₂-TiCl₄

- D1 blends → $\Delta\eta_{th} \approx 2-3pp$, D2 blends → $\Delta\eta_{th} \approx 0 - 1pp$
- Actually, $\Delta\eta_{th} < 0$ for some D2 blends (even if η_{th} slightly higher than D1 are achieved)



More info in papers submitted to **Turbo Expo 2021** and **ORC Munich 2021** (see last slide for more details)



Minimum cycle temperature depends on Ambient Temperature

PIT →	30°C	35°C	40°C	45°C	50°C	55°C	60°C
CO ₂	-	43.69	43.15	42.60	42.06	41.51	40.96
D1C10	50.96	50.21	49.36	48.61	47.64	47.03	45.94
D1C15	51.39	50.4	50.10	49.19	48.43	47.99	47.02
D1C20	50.50	50.25	49.97	48.76	48.41	48.19	47.05
D1C25	50.46	49.29	48.82	48.59	48.34	47.13	46.97
D2C15	53.47	52.98	52.45	51.89	51.14	50.30	49.42
D2C20	54.48	53.23	52.62	52.42	50.65	50.24	48.84
D2C25	52.54	51.35	50.73	49.64	48.34	47.13	46.97

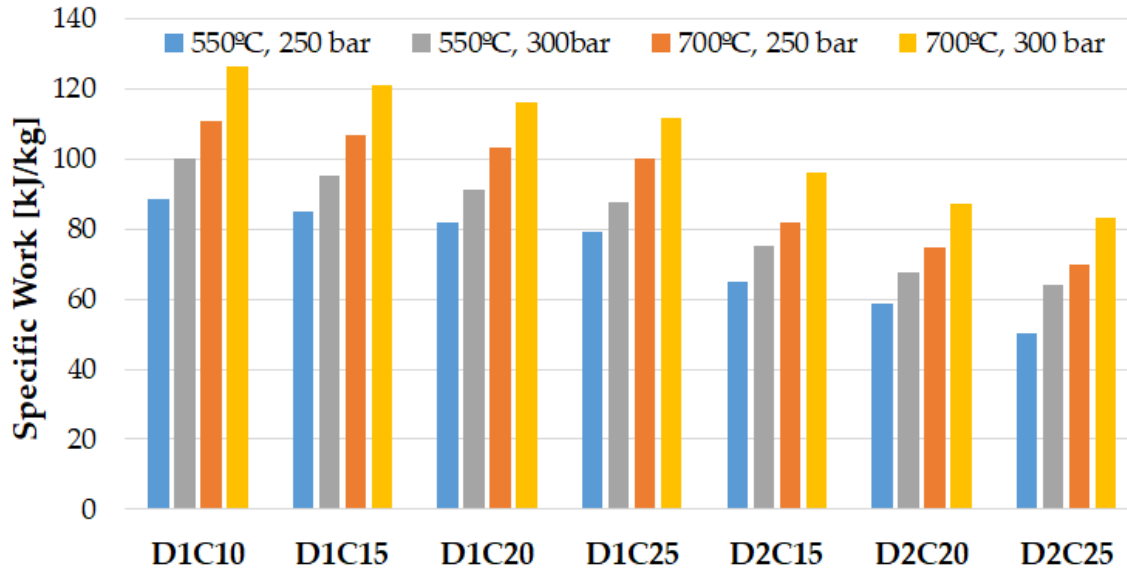
T_{min} is varied from 30 to 60°C, for 700°C/250bar case

Cycle considered: *Recuperated Brayton (sCO₂)*
Recuperated Rankine (blends)

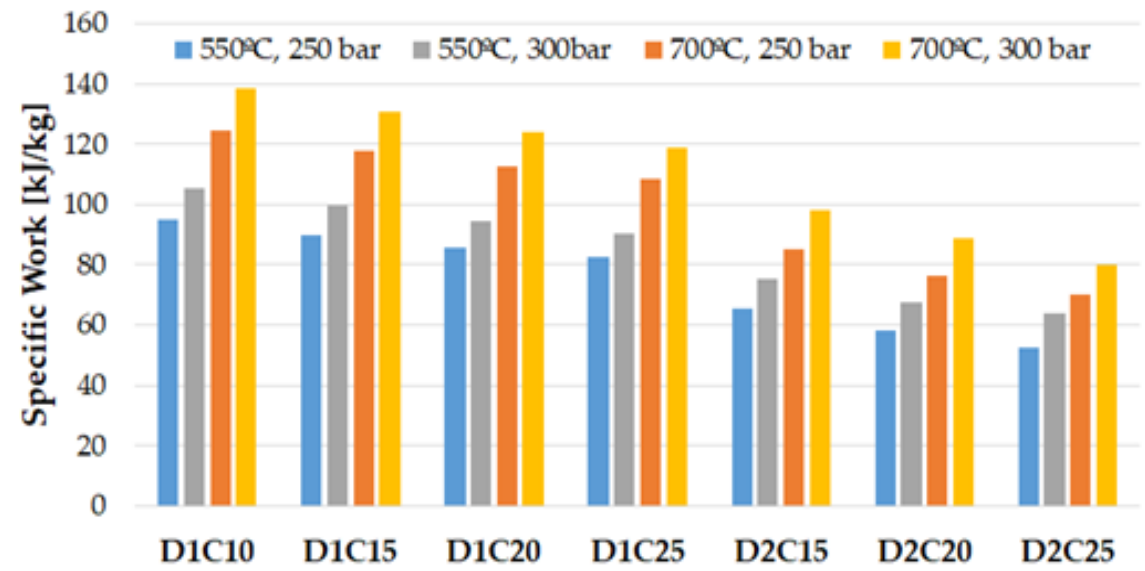
CO₂-blends prove to enable higher thermal efficiency than pure CO₂, employing a cycle with similar complexity.

Absolute best-performing blend regardless of minimum cycle temperature cannot be identified

- Possibility to tailor the composition of the working fluid to the ambient conditions of the plant site in order to maximise thermal performance
- Further optimisation of supercritical power cycles using CO₂ blends in other applications



Recuperated Rankine



Precompression

- *Precompression* yields higher W_s than *Recuperated Rankine* (higher expansion ratio brought by compressor)
- Similar patterns: smallest molar fraction \rightarrow highest W_s (for a given set of boundary conditions)
- For a given dopant: \uparrow molar fraction $\rightarrow \uparrow$ molar weight $\rightarrow \uparrow$ circulating mass flow $\rightarrow \downarrow W_s$
- D2 present W_s lower than D1 blends \rightarrow a compromise is needed between these two figures of merit



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- The use of CO₂-based mixtures in supercritical cycles enables $\eta_{th} \geq 50\%$, even for $T_{min} \geq 50^\circ\text{C}$. This is well above what state-of-the-art Rankine cycles running on steam are currently achieving in CSP plants → significant upsurge in performance for this type of application.
- CO₂-based mixtures also enable thermal performance much better than what conventional sCO₂ can attain for the same boundary conditions (due to the deleterious effect of high T_{amb} on the compression process). The gain enabled CO₂-based mixtures is in the order of 5-6 pp ($\geq 10\%$ relative performance improvement with respect to an equivalent embodiment with pure CO₂).
- The performance of supercritical cycle layouts using CO₂-based mixtures depends weakly on turbine inlet pressure, whereas the influence of minimum and maximum cycle temperatures is very strong. Nevertheless, regarding temperature, whilst both temperatures determine thermal efficiency, turbine inlet temperature does not have any influence on the composition of the blend yielding the best performance. This seems to be dependent on minimum cycle temperature only.
- When CO₂-based mixtures are used, cycle optimisation must include WF composition and cycle layout as independent variables. This is because the layout yielding the best η_{th} changes as mixture composition changes.



Thank you for your attention!
Any Questions?





- F. Crespi, G.S. Martínez, P.R. de Arriba, D. Sánchez, F. Jiménez-Espadafor, *Influence of Working Fluid Composition on the Optimum Characteristics of Blended Supercritical Carbon Dioxide Cycles*, **ASME Turbo Expo 2021: Turbomachinery Technical Conference and Expositions** (June 2021)

- F. Crespi, P.R. de Arriba, D. Sánchez, A. Muñoz, T. Sánchez, *The Potential of Supercritical Cycles Based on CO₂ mixtures in Concentrated Solar Power Plants: An Exergy-based Analysis*, **ORC Munich 2021: 6th International Seminar on ORC Power Systems** (October 2021)



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