

Development and Testing of Dry Gas Seals for Turbomachinery in Multiphase CO₂ Applications

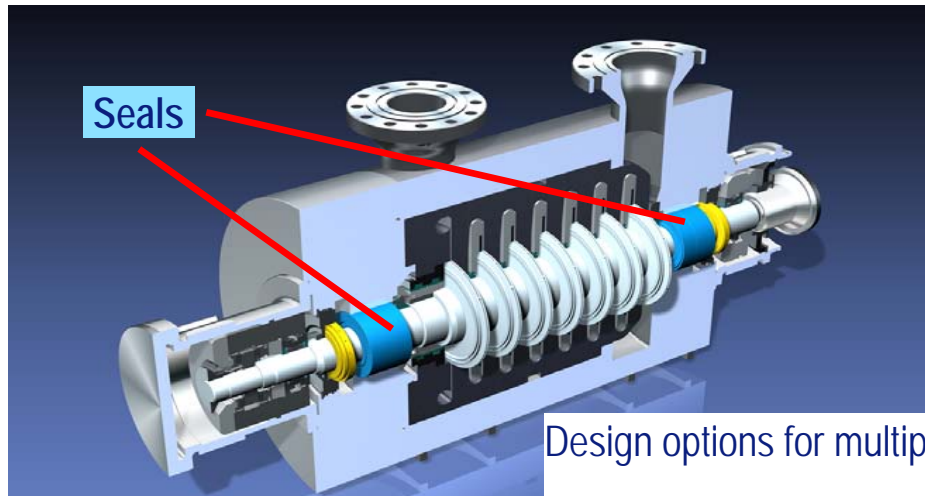
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Multiphase Seals in Turbomachinery Applications

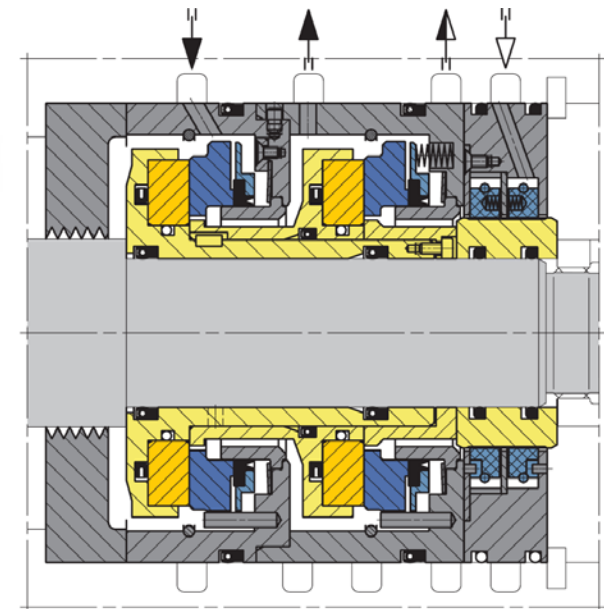
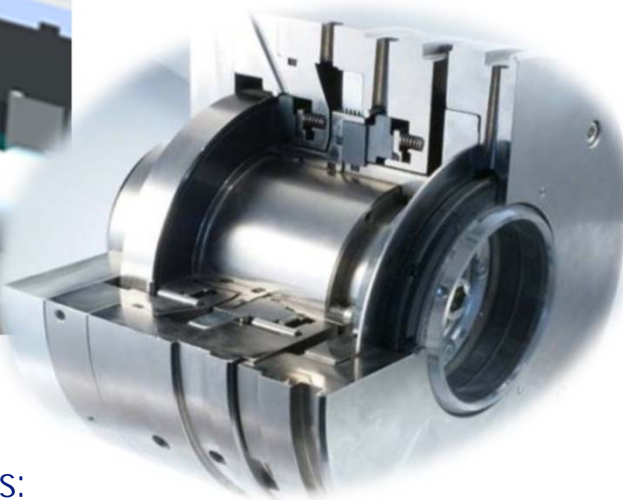
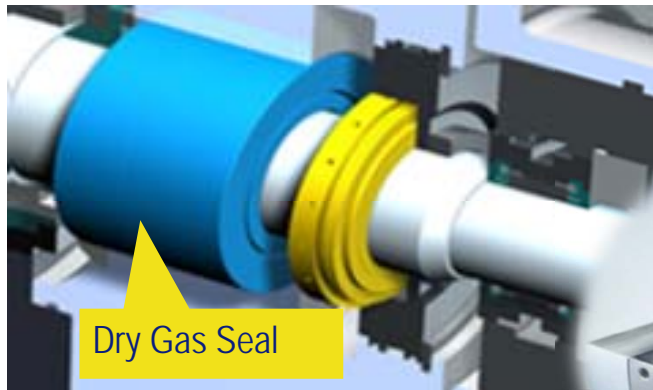


Low leakage demands require **mechanical seals!**

Design options for multiphase applications:

- A) Liquid Seal contacting seal, small sealing width, low speed limits, low liquid leakage, cannot work with pure gas
- B) Dry Gas Seal non-contacting, large sealing width, material determined speed limits, can work with liquids (in a limited range), very low gas leakage, high liquid leakage
- C) Liquid/Gas Seal features of both seal designs are combined

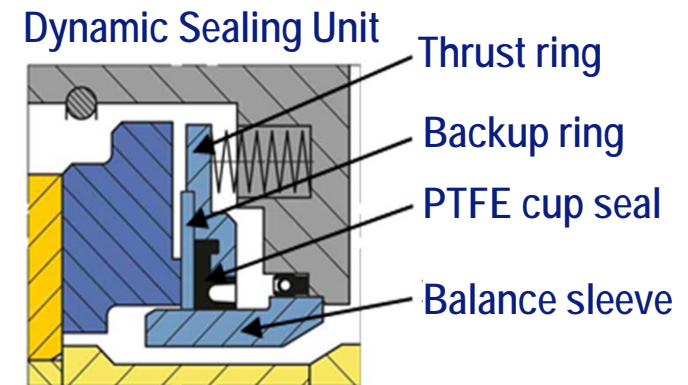
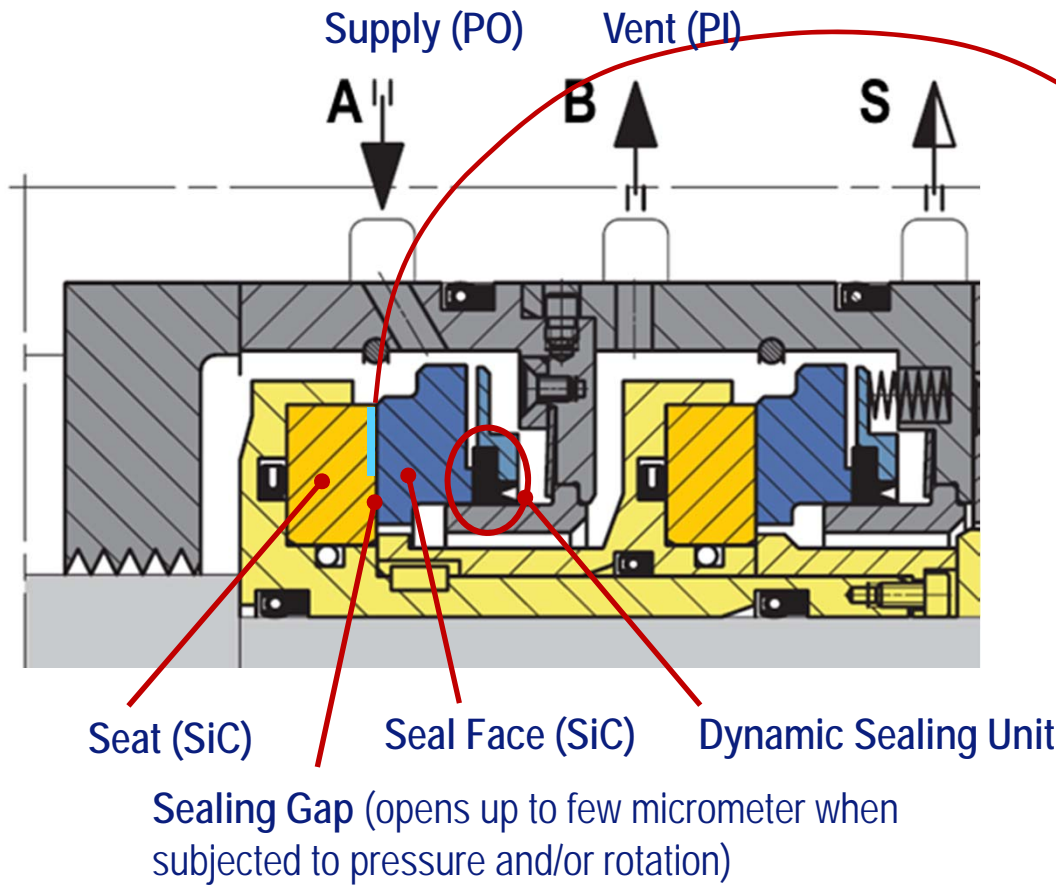
Multiphase Seal



Dry Gas Seal Characteristics:

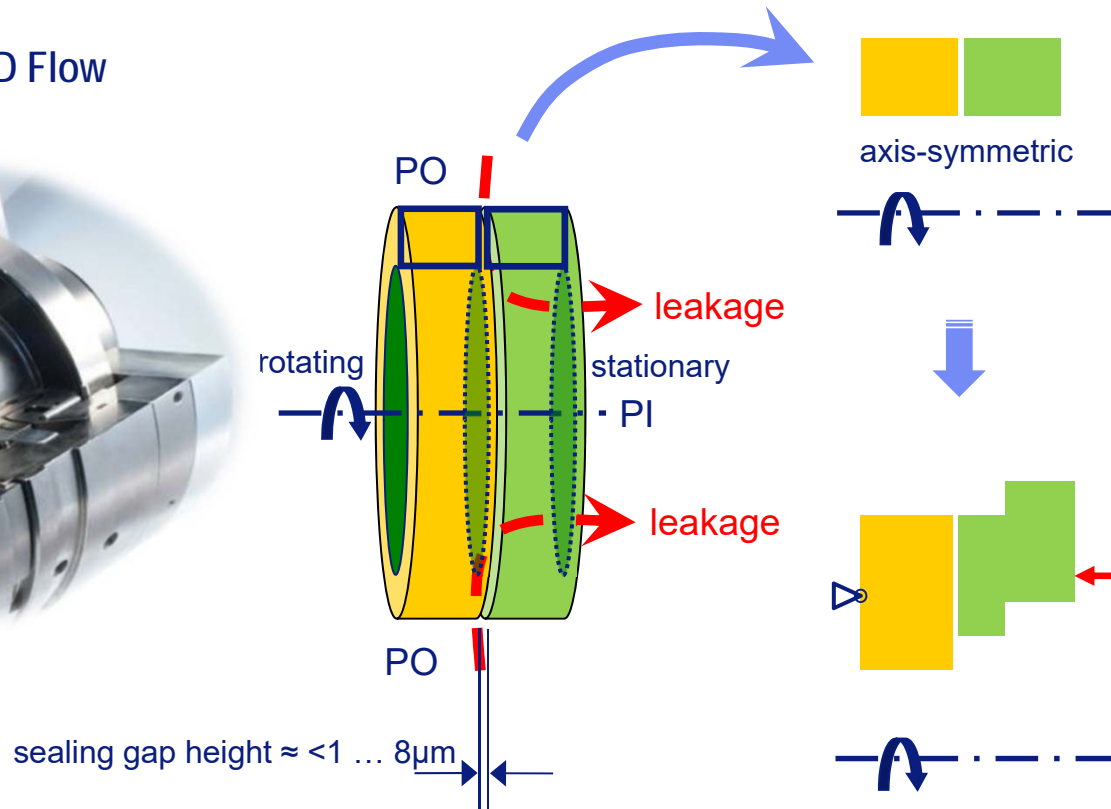
- Low gas leakage
- Low friction losses
- No wear in sliding face

Dry Gas Seal (DGS) – Design

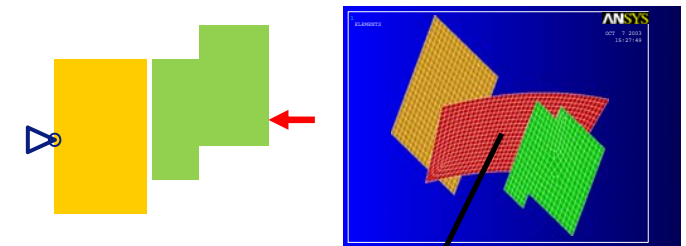


Dry Gas Seal – Analysis

3D Structure / 3D Flow



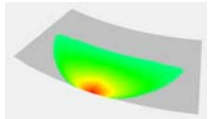
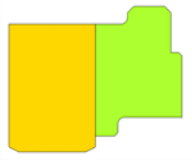
2D Structure / 2D Flow



$$\frac{1}{r} \frac{\partial}{\partial \theta} \left(\frac{\rho h^3}{12\eta} \frac{\partial p}{\partial r} \right) + \frac{\partial}{\partial r} \left(\frac{r \rho h^3}{12\eta} \frac{\partial p}{\partial r} \right) = \frac{r\omega}{2} + \frac{\partial}{\partial \theta} (\rho h)$$

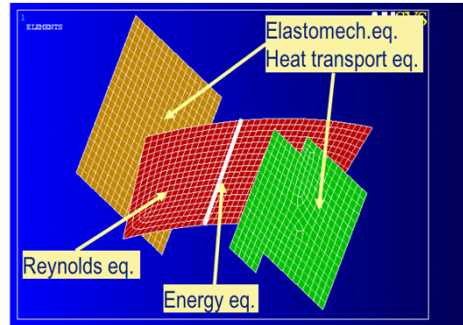
Reynolds Equation

Sealanalysis – Coupled Fluid-Structure-Analysis



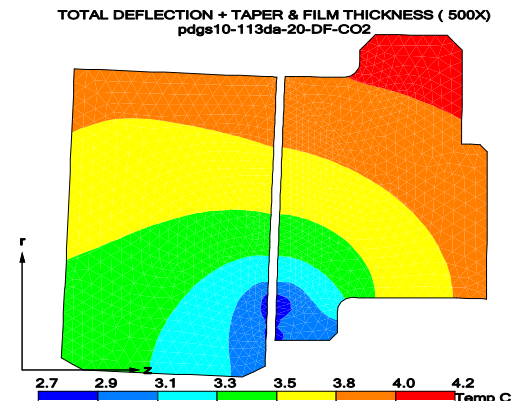
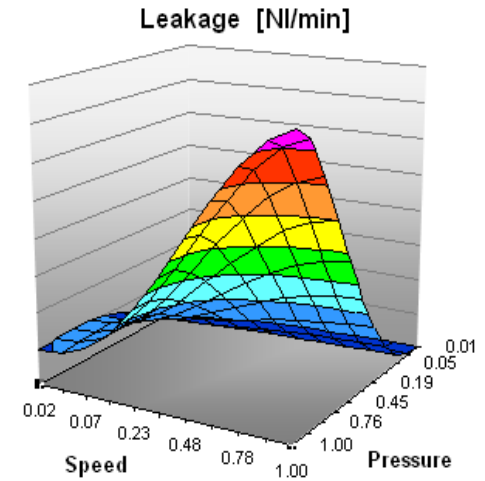
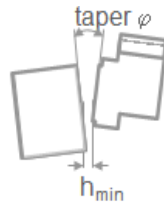
Design Data
Operating Data

Fluid
Properties



SEALANALYSIS
Additional Computational Design Tools

Gap flow: p, h, v cyclic symmetric, $f(r, \theta)$
 Interface: T, q $f(r)$
 Ring System: T, u, q axis-symmetric, $f(r, z)$



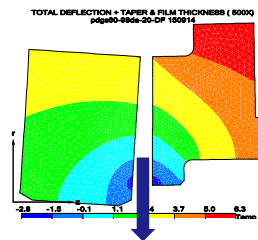
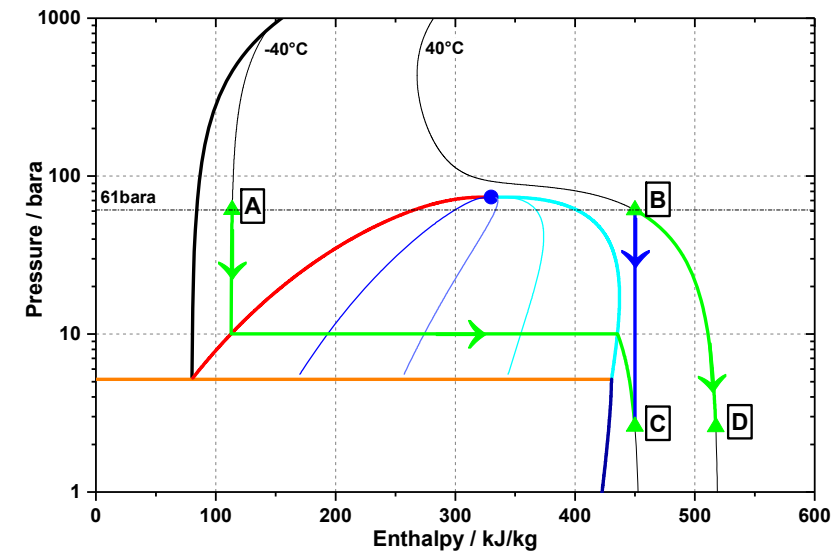
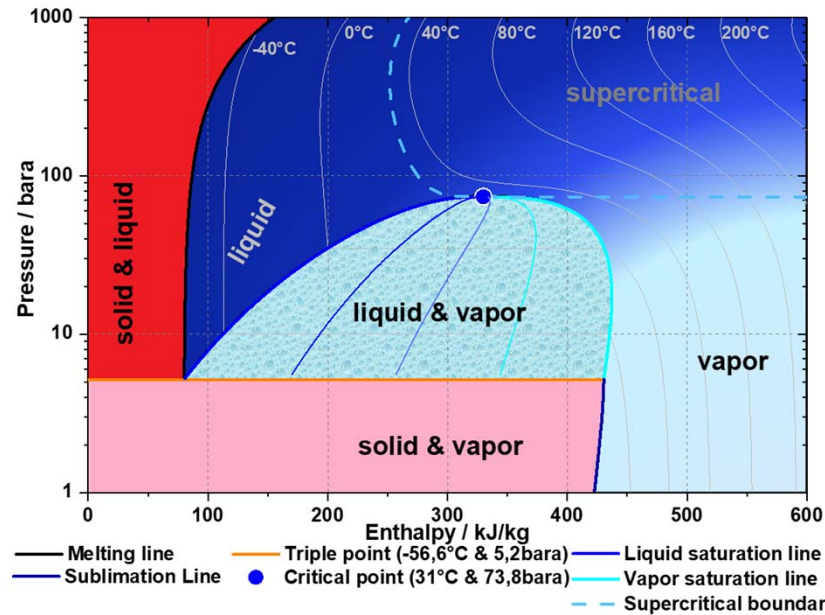
DGS for CO₂ Multiphase Applications

What's different with CO₂ compared to most other fluids?

- Gas properties, in particular in transcritical region.
- Operation without additional heater possible?
- Are numerical predictions correct?

=> **Verification / validation of design by internal test campaign!**

Criticality of CO₂ Sealing Applications



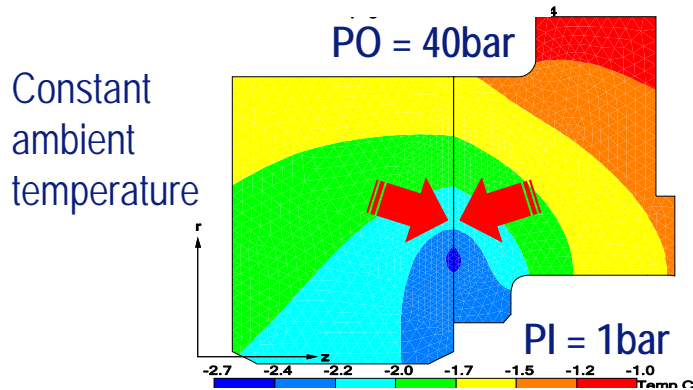
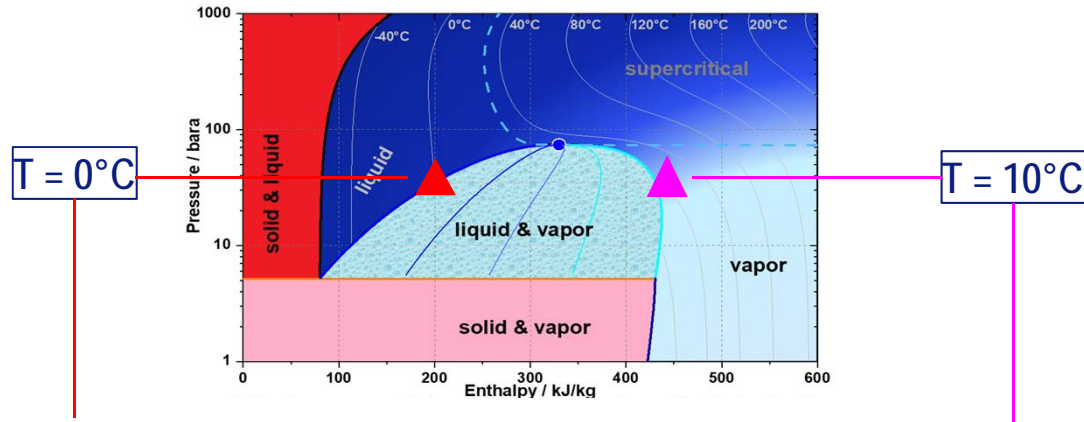
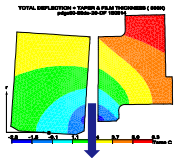
A → C ... isothermal, $\Delta T = 0K$

B → C ... isenthalpic, $\Delta T = -80K$

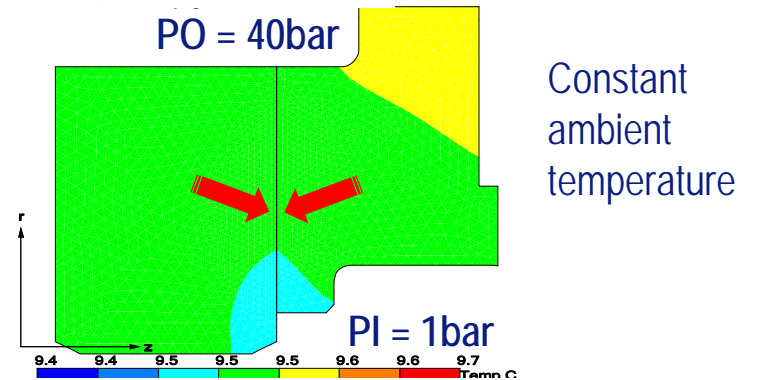
B → D ... isothermal, $\Delta T = 0K$

(Almost) Isothermal Expansion – What does it really mean?

PO = 40bar, PI = 1bar
CO₂, Static



Expansion with only little temperature drop in the ring system is referred to as **almost isothermal** or simply **isothermal!**



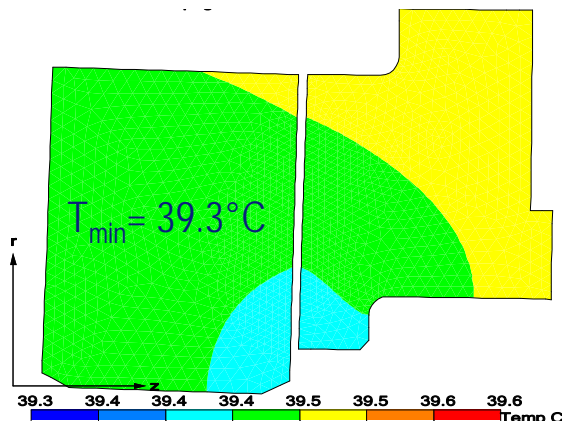
$T \gg -56.6^\circ\text{C}!$

Criticality of CO₂ Sealing Applications

PO = 60barg, PI = 0barg, **T = 40°C**, CO₂, Static

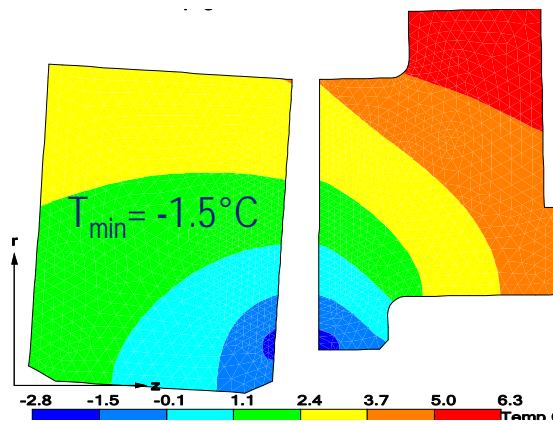
Result plots: Total deflection and isotherms (scale 500X)

Reference



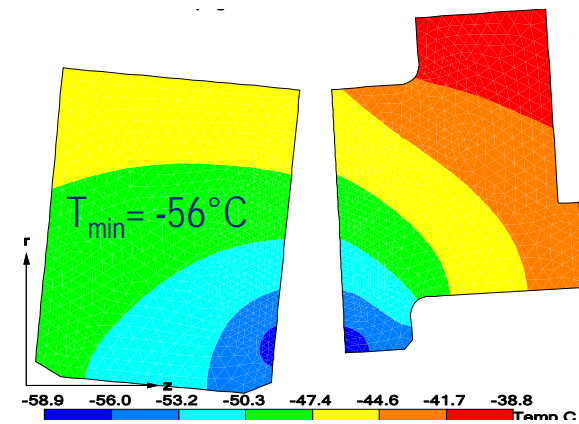
Leakage: 2.2 NI/min

Inappropriate gap design 1



107 NI/min

Inappropriate gap design 2

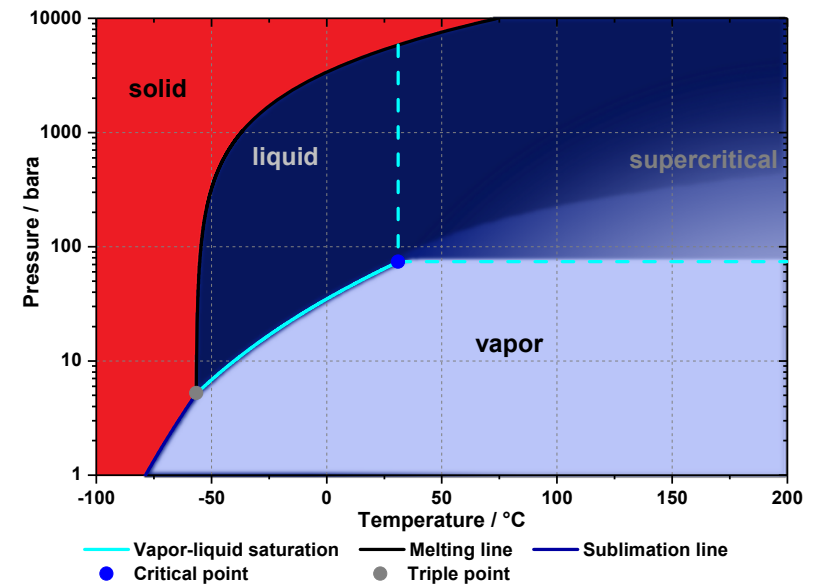


260 NI/min

Avoiding adverse effects on seal performance requires extremely low leakage levels!

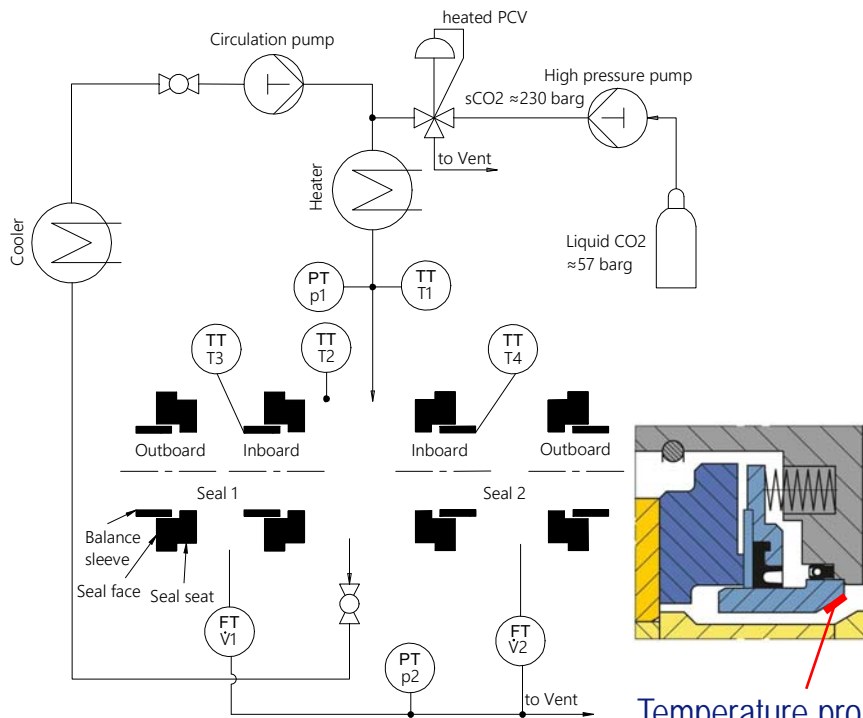
Test Set-up and Testing Range (Pressure/Temperature)

Multiphase suitability: Seal can operate in liquid, two-phase, vapor and supercritical region **without additional heater!**



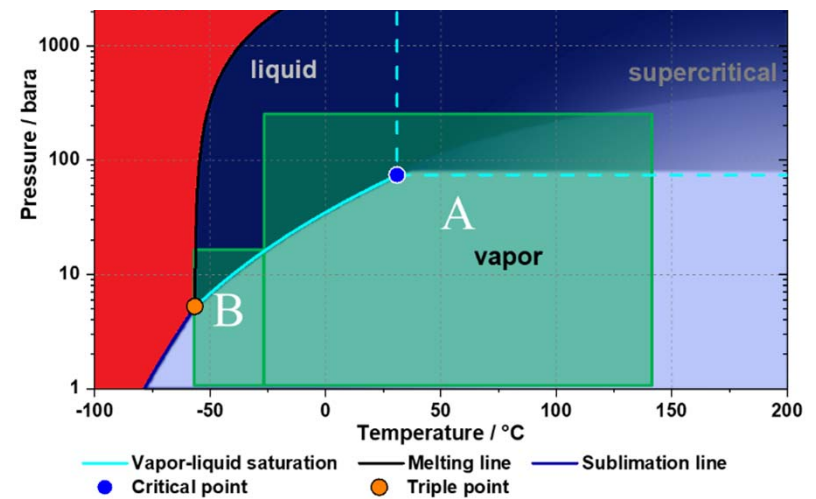
Test Set-up and Testing Range (Pressure/Temperature)

Simplified PID



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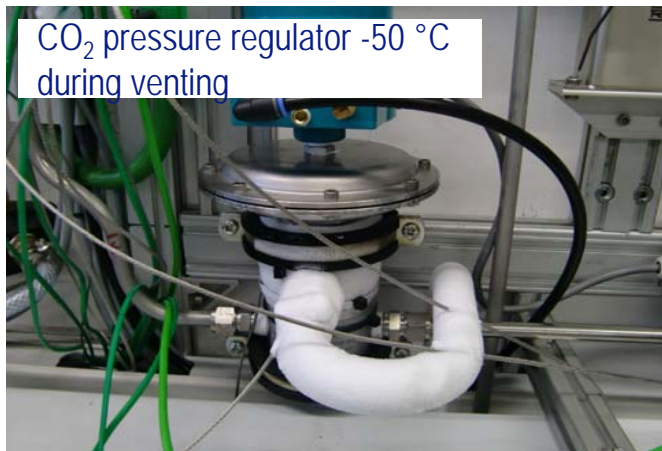
P-T Range of Testing



A ... normal operating range

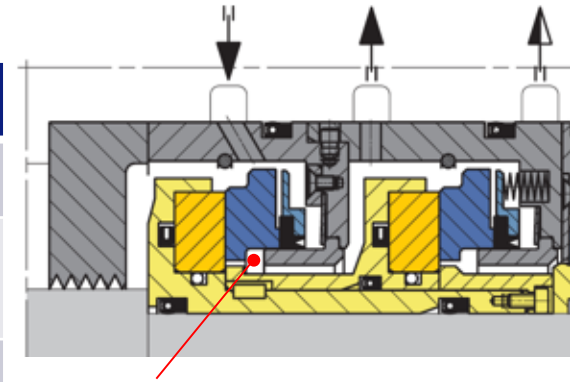
B ... failure test

CO₂ Test Facility



Test Program

		Pressure	Speed	Temperature	Time
		barg	rpm	°C	h
Tandem seal	static	60 / 80 / 200	-	0 ... 120	
	static	60	-	0 ... 100	18
	dynamic	60 / 80 / 180	3600	0 ... 120	
Single seal	static	60	-	ambient	48
	static	200	-	ambient	14



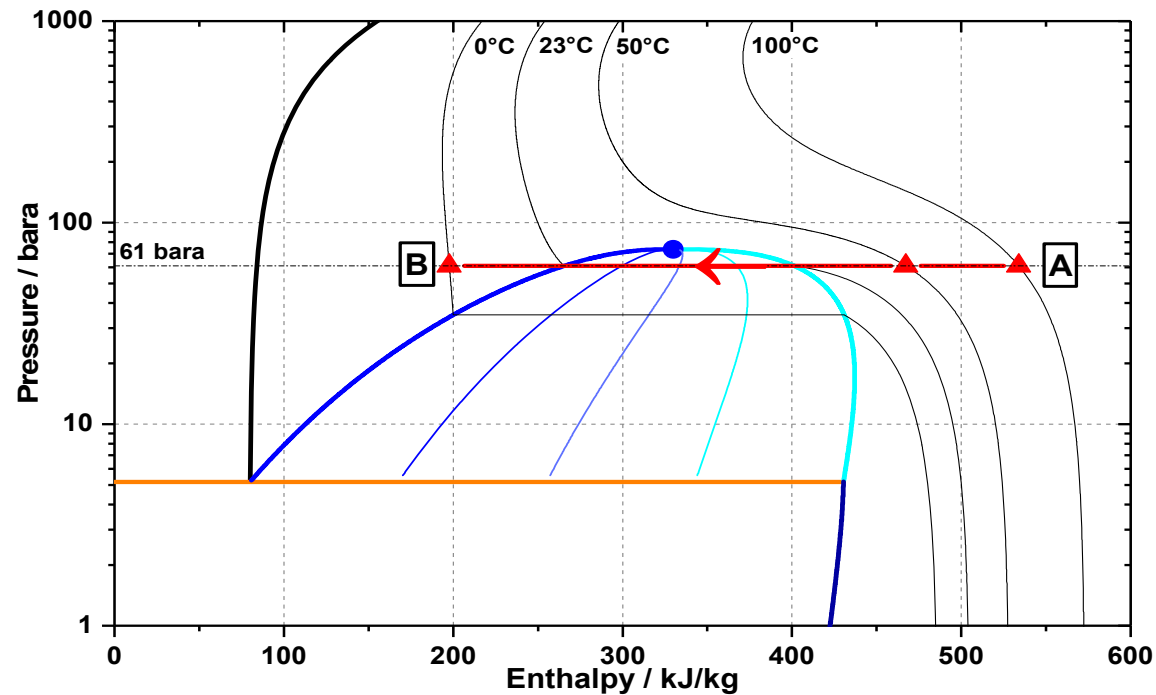
Low pressure area:

- optional protected by N₂ (tandem seal)
- exposed to ambient conditions (single seal)

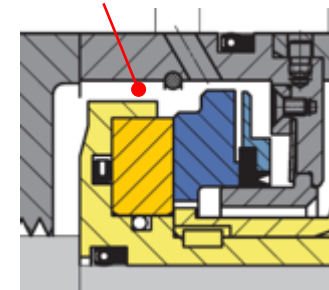
Additional: Seal failure simulation

60barg CO₂ Static Test, Decreasing Temperature

Test section: Cool down phase (18h) after dynamic run with hot CO₂



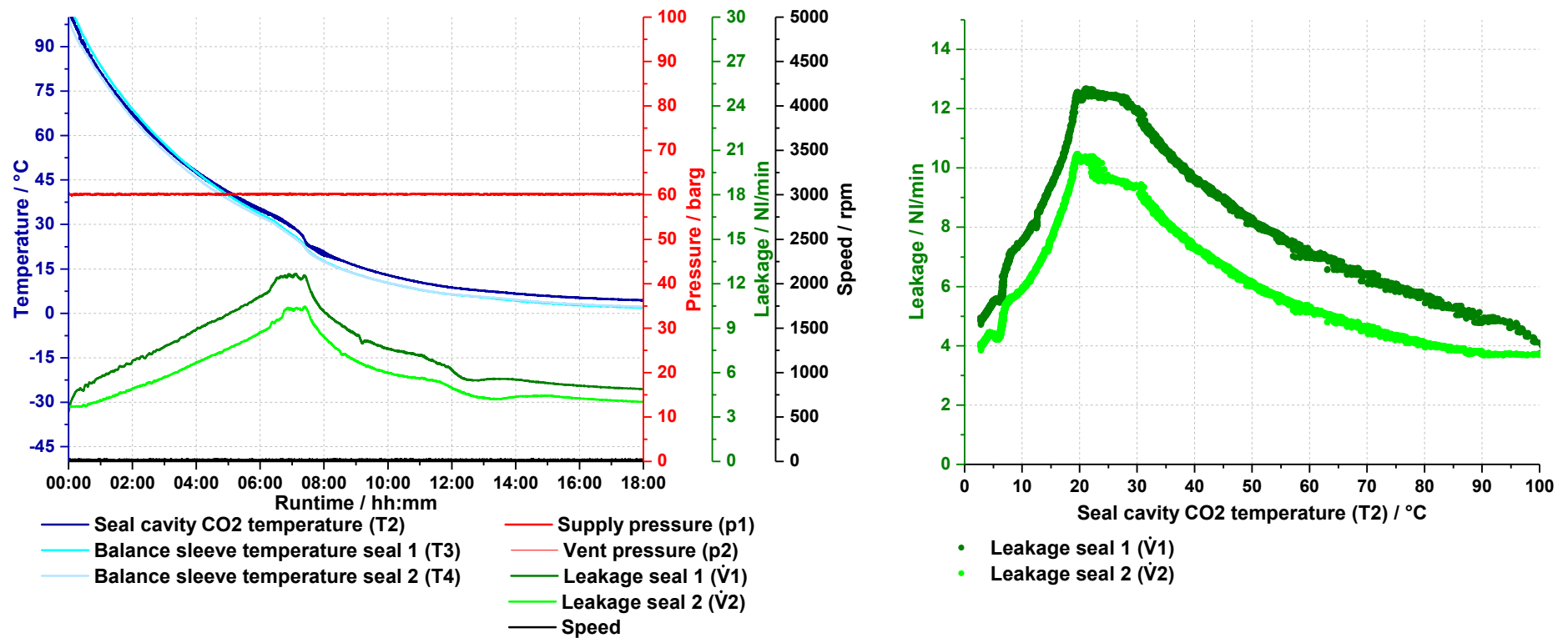
P const., T decreasing



- A** Start in gaseous region
60barg/100°C
- B** End in liquid region
60barg/0°C

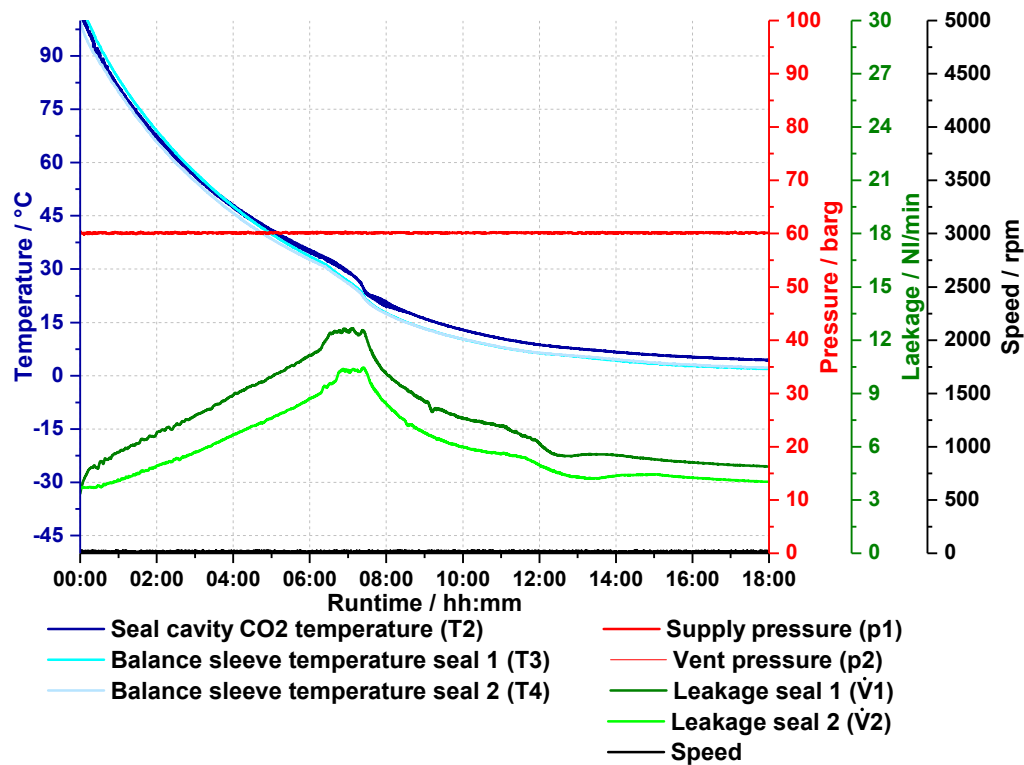
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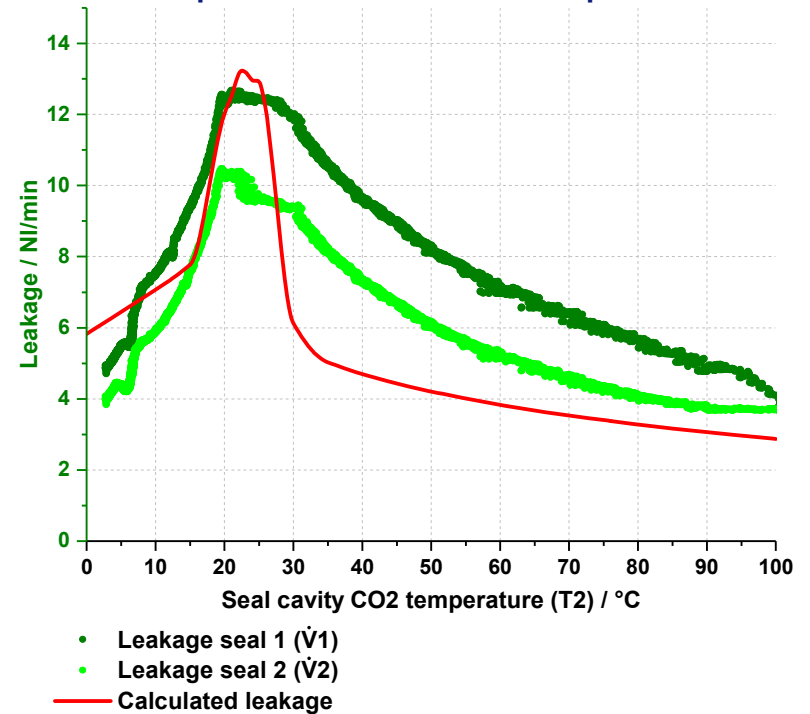


60barg CO₂ Static Test, Decreasing Temperature

Test section: Cool down phase (18h) after dynamic run with hot CO₂

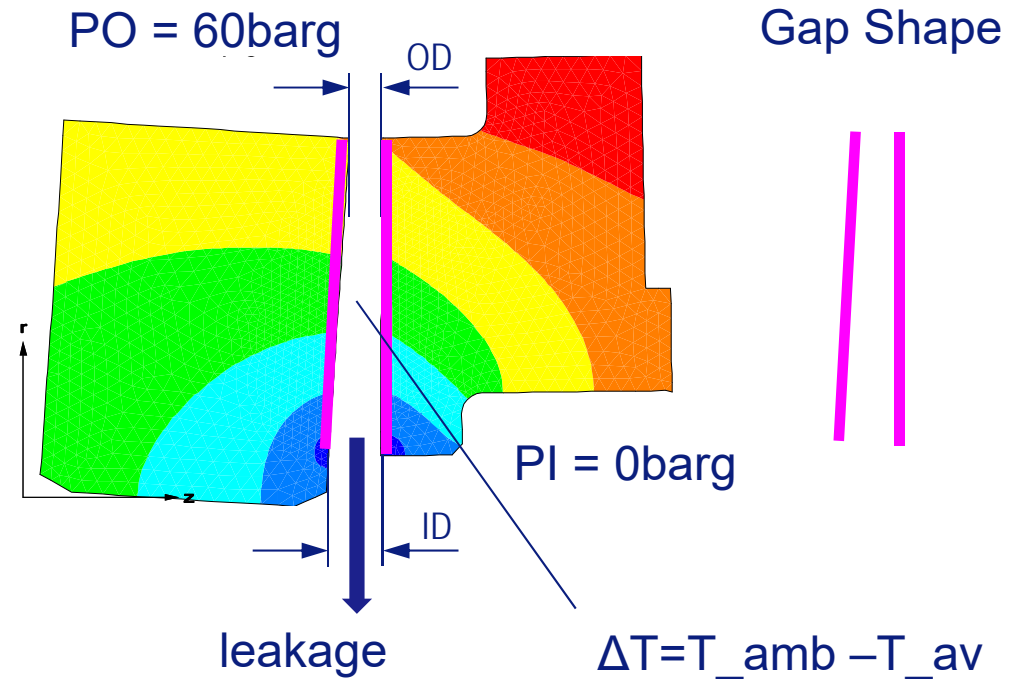
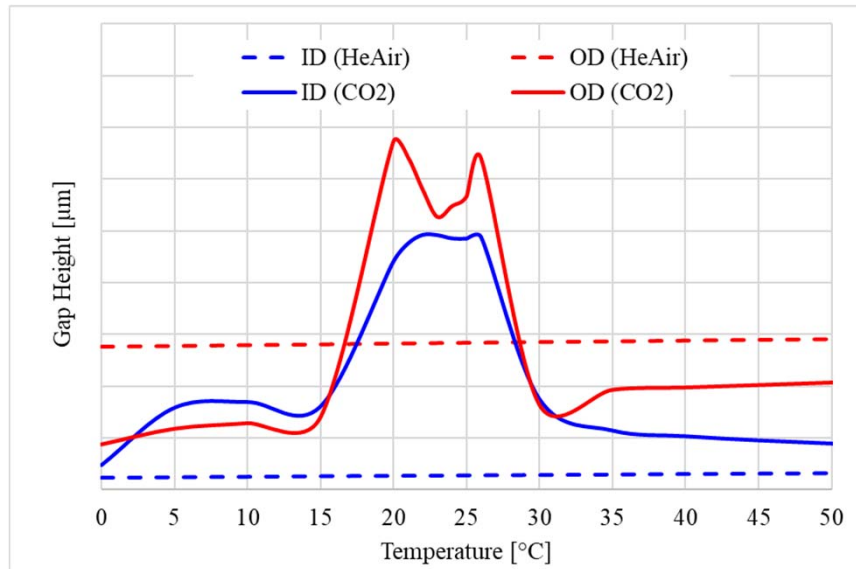


Comparison with numerical prediction



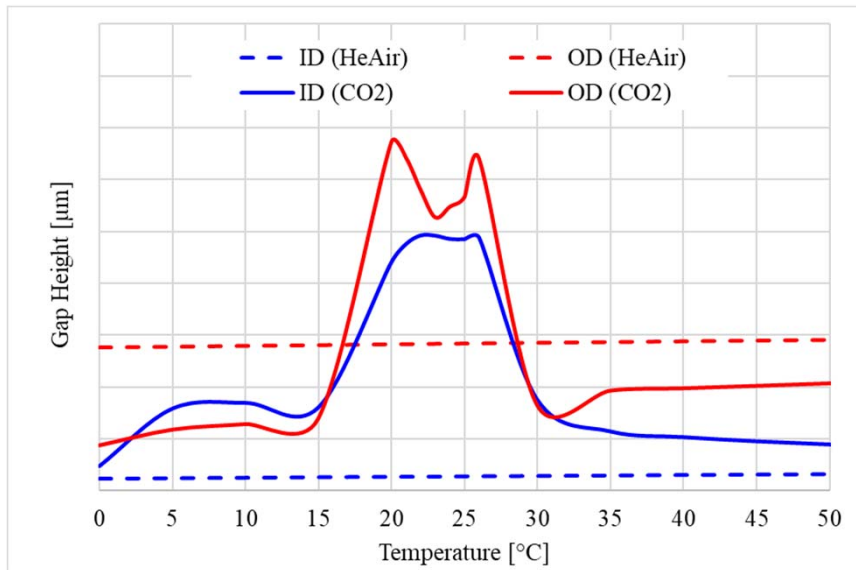
60barg CO₂ Static Test, Decreasing Temperature – Analysis

Numerical prediction of gap shape, leakage and temperature

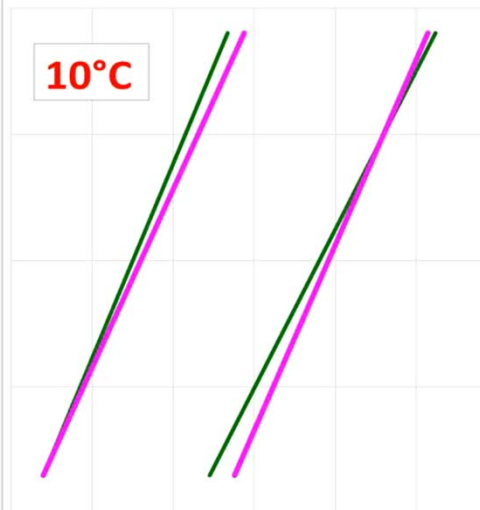


60barg CO₂ Static Test, Decreasing Temperature – Analysis

Inner/outer height of sealing gap

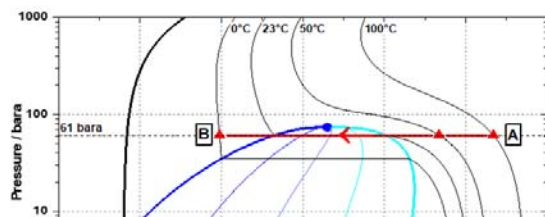
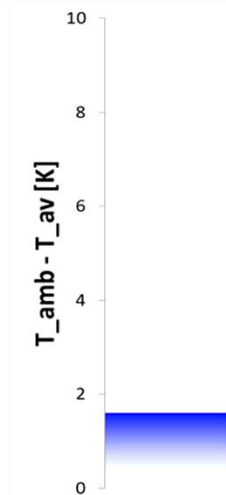
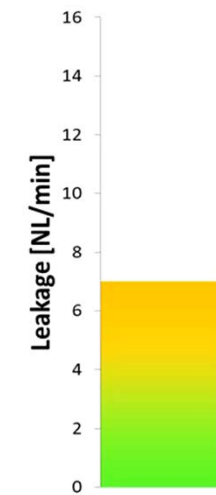


Gap shape

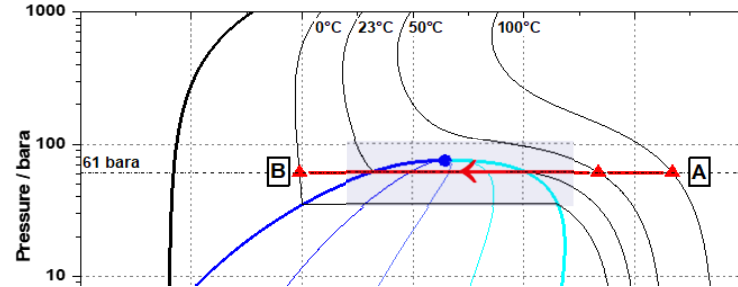
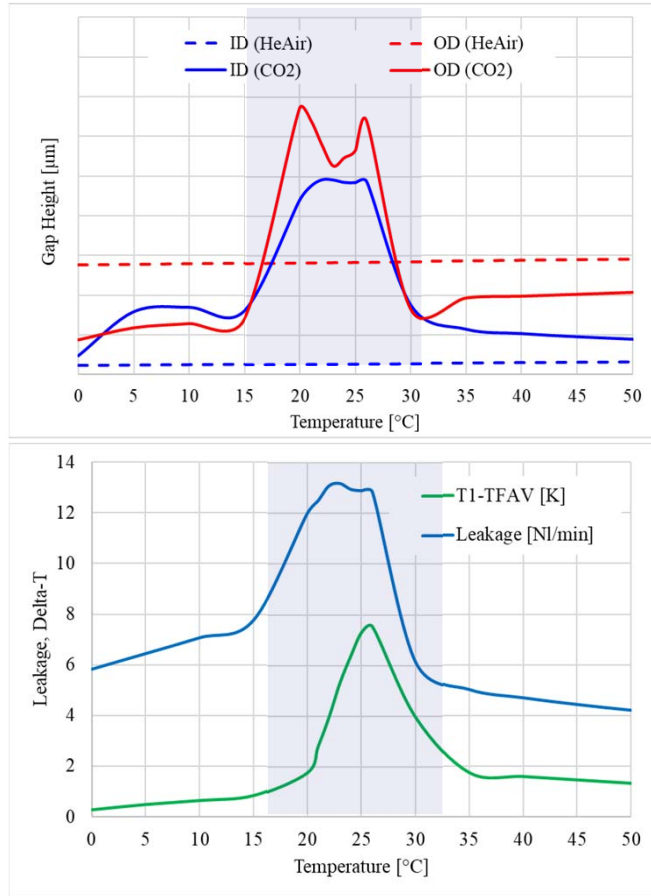


\dot{V}

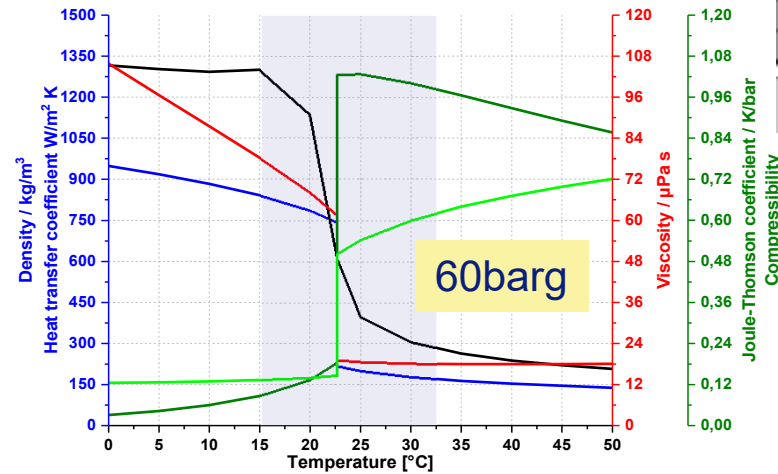
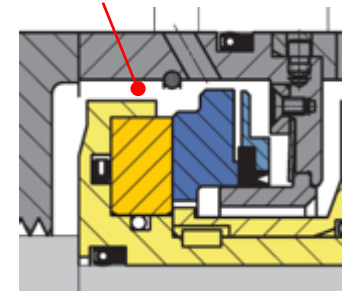
ΔT



60barg CO₂ Static Test, Decreasing Temperature – Analysis



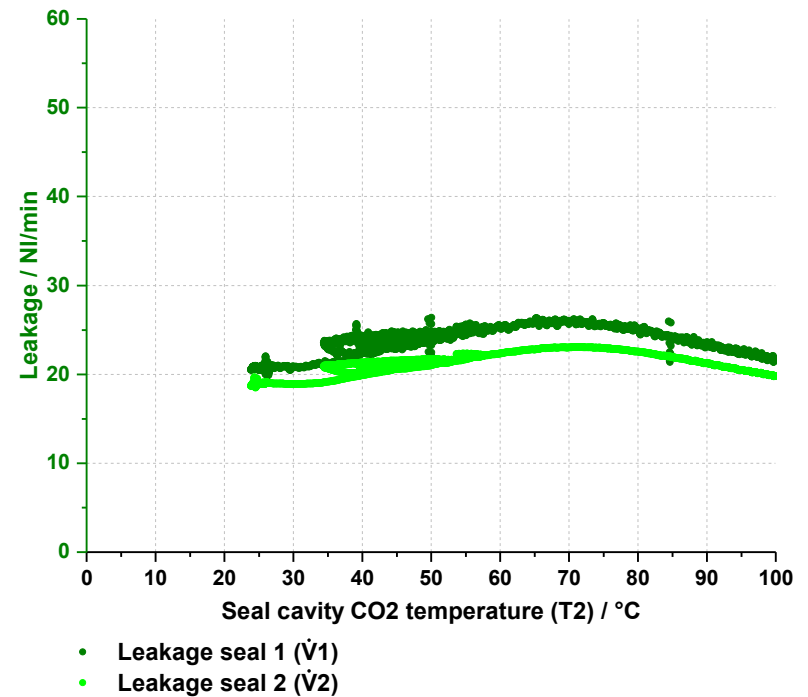
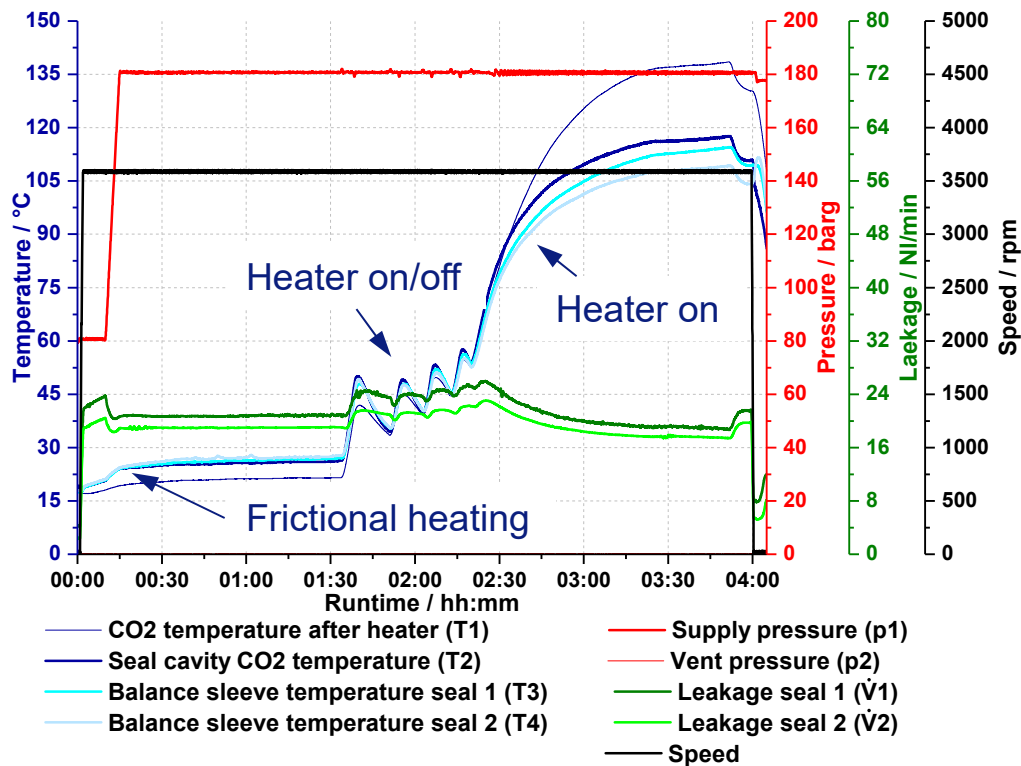
P constant
 $P_{ar} = F(T)$



- Density
- Heat transfer coefficient
- Viscosity
- Joule-Thomson coefficient
- Compressibility

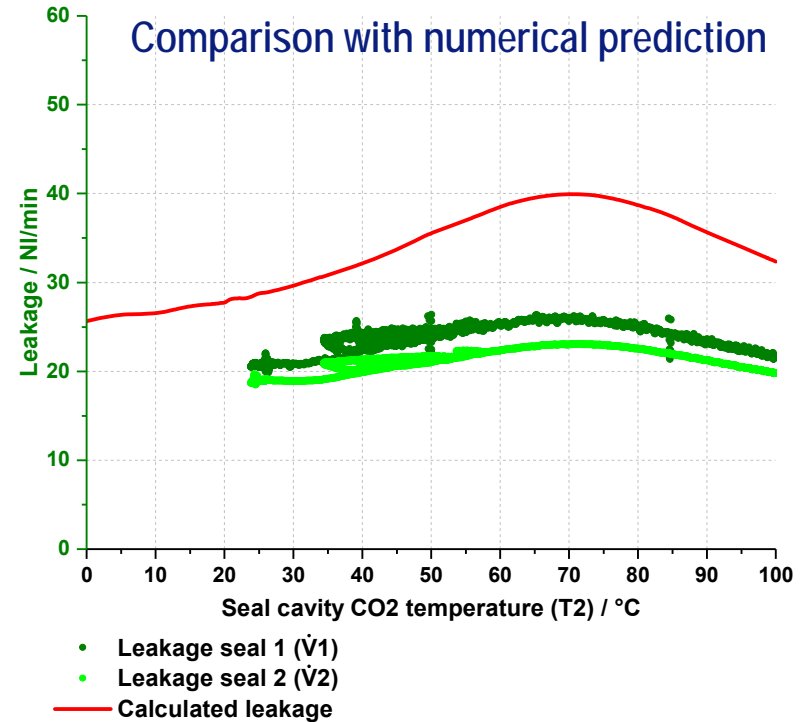
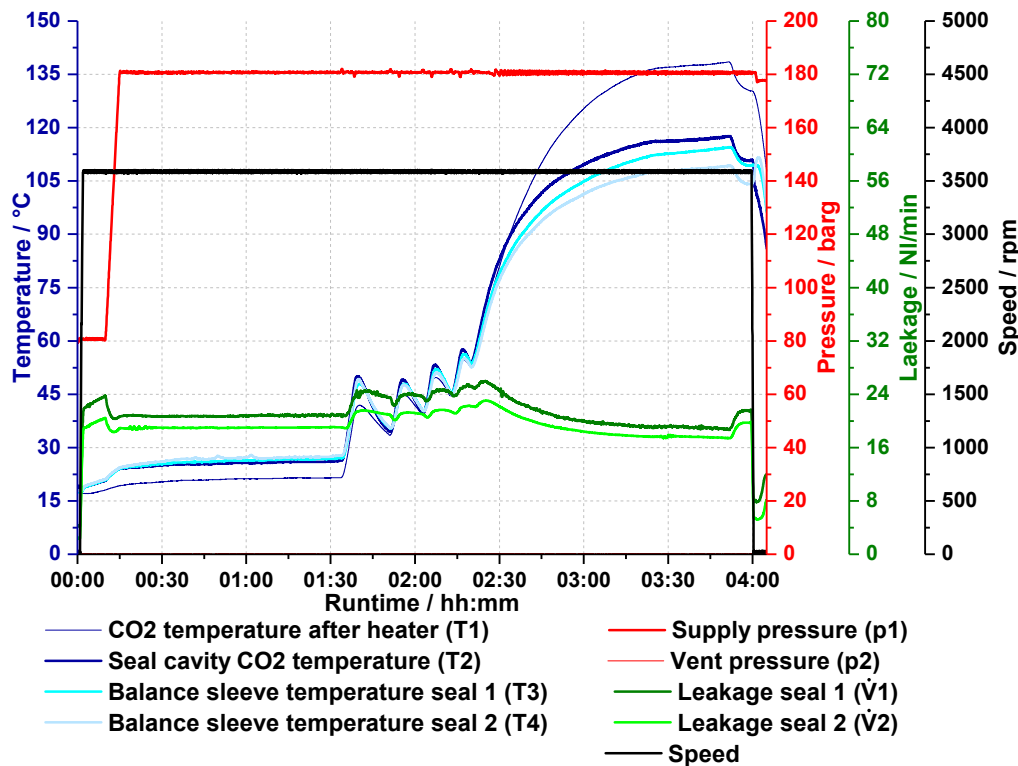
180barg CO₂ Dynamic Test, Increasing Temperature

Test section: Dynamic run (180barg/3600rpm), CO₂ temperature control: 20°C / 50°C / 120°C

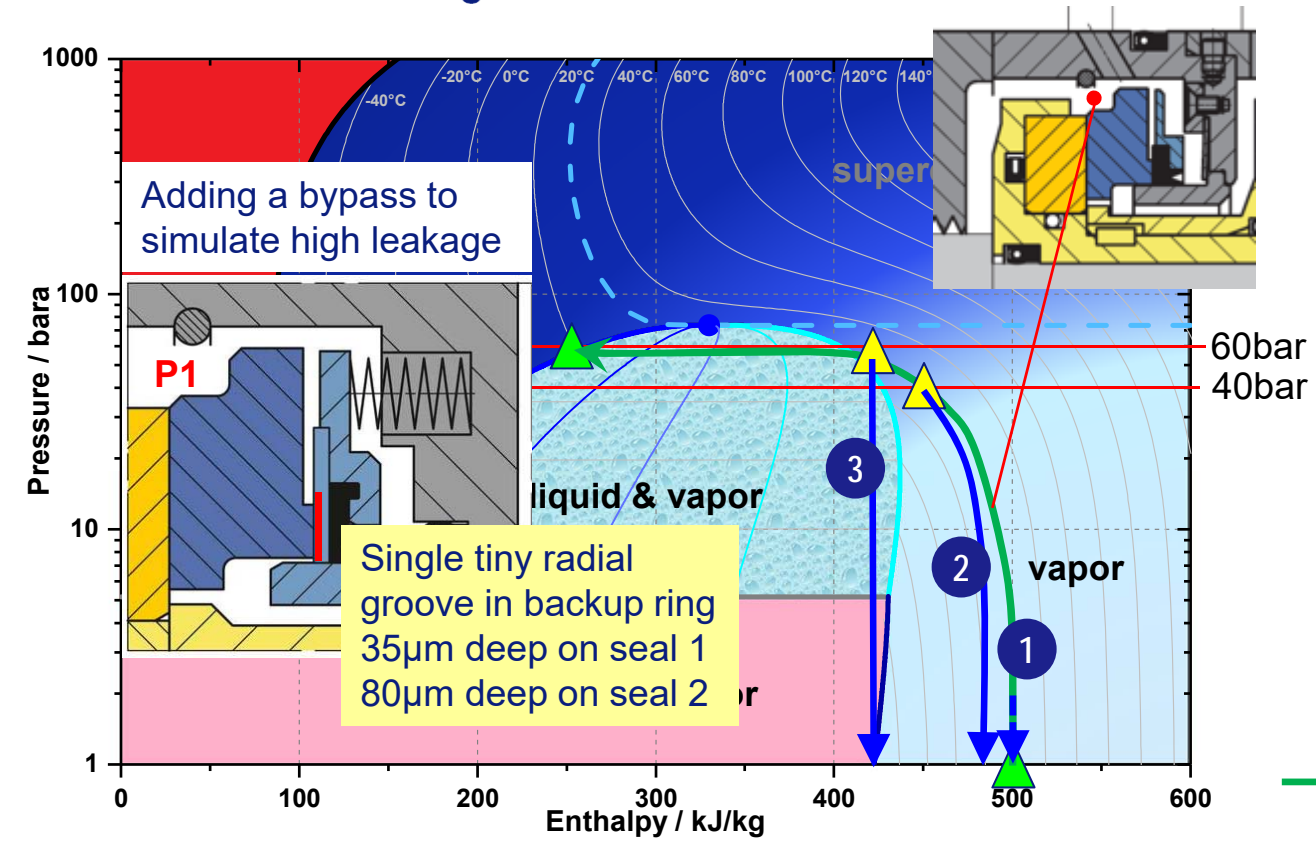


180barg CO₂ Dynamic Test, Increasing Temperature

Test section: Dynamic run (180barg/3600rpm), CO₂ temperature control: 20°C / 50°C / 120°C



Static Test Simulating a Seal Failure



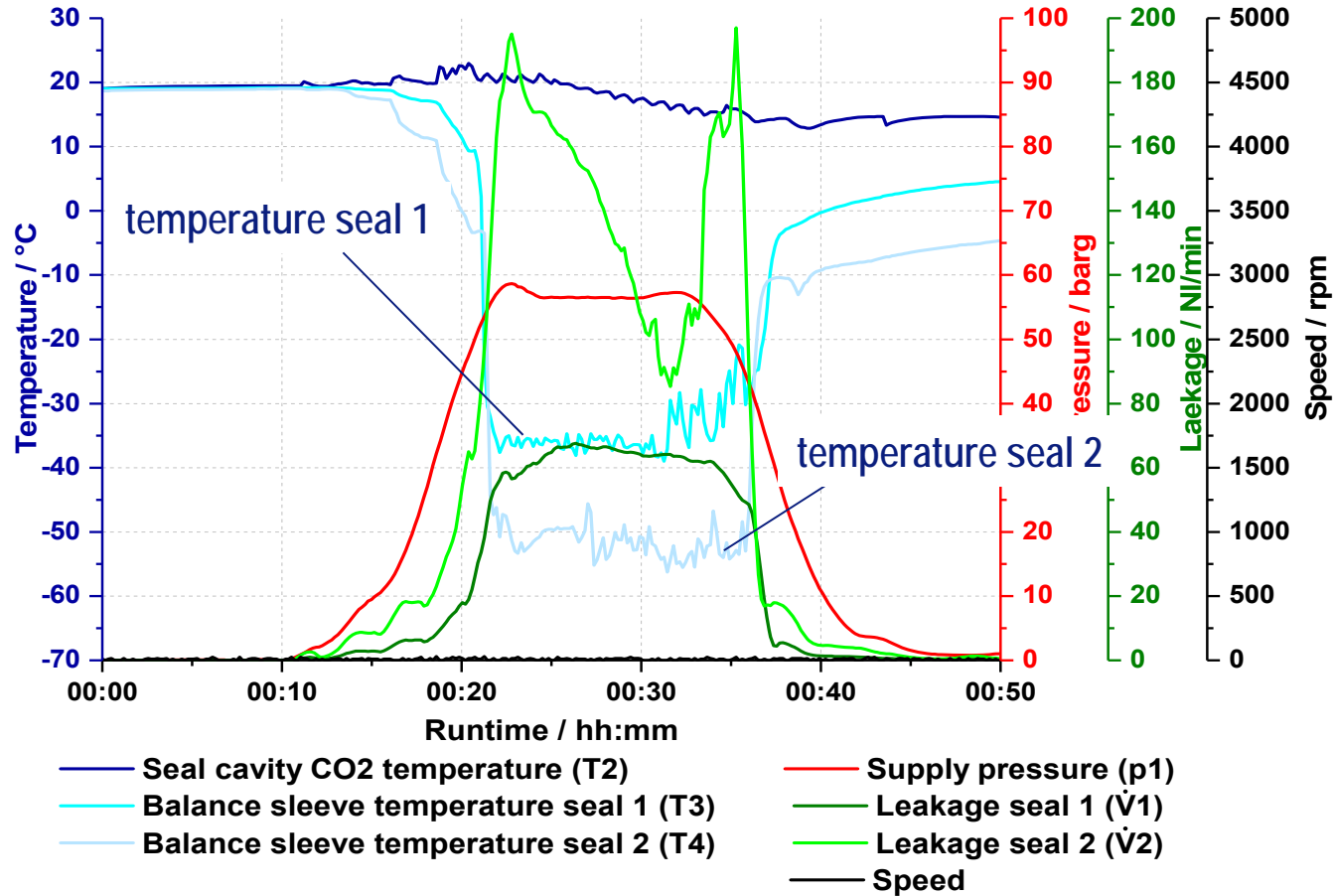
- 1 Sealing cavity pressure increase at constant temperature 20°C
- 2 At about 40bar remarkable deviations from isothermal expansion. Seal temperature about 0°C
- 3 Beyond 55bar isenthalpic expansion down to triple point

→ Sealing cavity pressure PO

→ Expansion through sealing gap & bypass

— Melting line — Triple point (-56,6°C & 5,2bara) — Liquid saturation line
 — Sublimation Line ● Critical point (31°C & 73,8bara) — Vapor saturation line
 — — Supercritical boundary

Static Test Simulating a Seal Failure – Results



Summary

- Dry Gas Seal design
 - large seal width provides good lift-off capability and thus wear-free operation
 - Safe non-contacting dynamic operation
 - Offers modification for compressor applications
- Virtually isothermal expansion due to extremely low leakage
- Works with liquid, two-phase, gaseous and supercritical CO₂ without the need for additional heating of the high-pressure sealing cavity
- Sound numerical predictive capability even close to critical CO₂ conditions and phase transitions

Rely on excellence

**Thank you
for your attention!
Questions?**

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Germany



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Dry Gas Seal (DGS) – Basic Components

